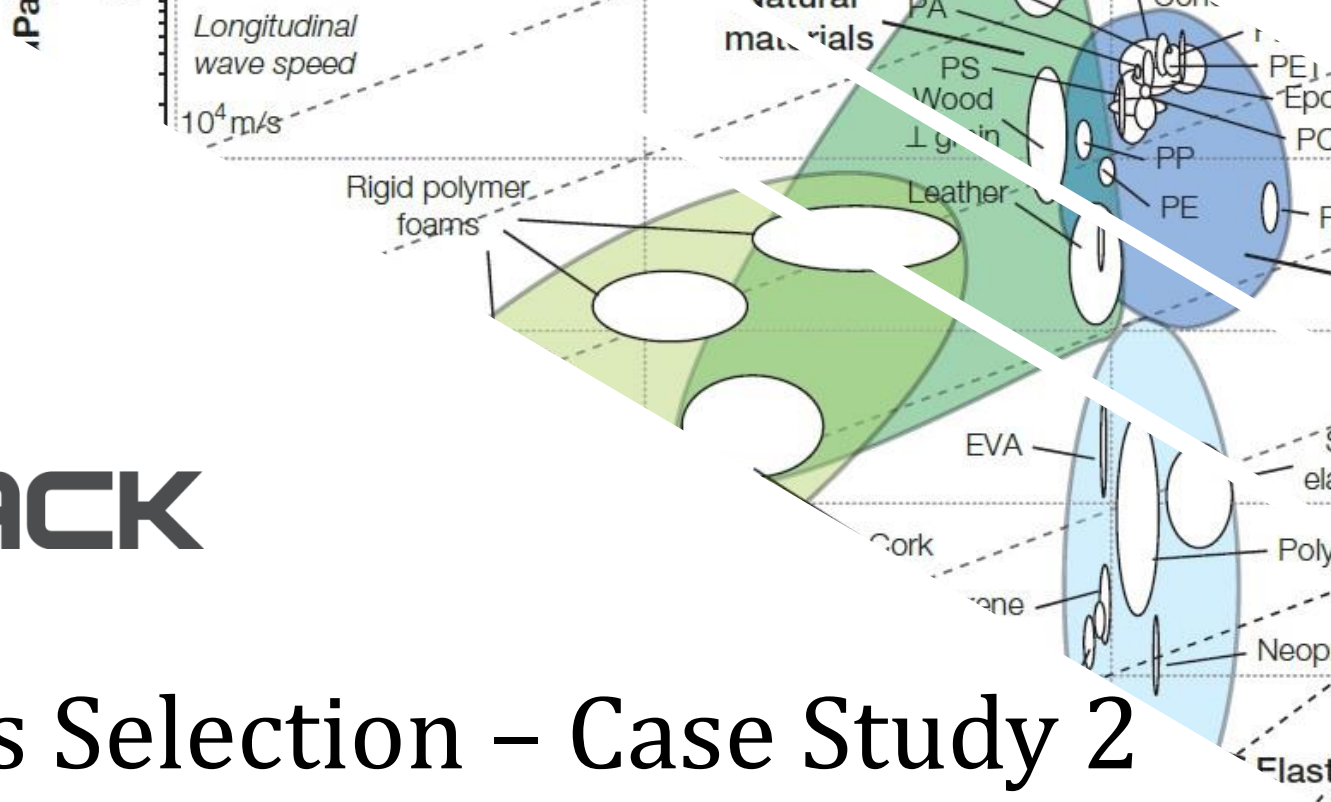




GRANTA
CES
EDUPACK



Materials Selection – Case Study 2

Advance Properties and Concepts

Professors:

Anne Mertens and Davide Ruffoni

Assistant:

Tommaso Maurizi Enrici



Outline

- ***Case Study 13: Load-limited design, Energy-limited design and Deflection-limited design***

- ***Case Study 14: Safe Pressure Vessels***

Multiple constraints

- ***Case Study 14': Light pressure Vessels with **MULTIPLE CONSTRAINT APPROACH*****

Theory: Method of the weight factor

Theory: Enhanced Digital Logic (EDL)

- ***Case Study 15: Precision devices***
- ***Case Study 16: Long Span Transmission line***
- ***Case Study 17: Light Cable***
- ***Case Study 18: Kiln Walls***
- ***Case Study 19: Insulation for Short-Term isothermal containers***
- ***Case Study 20: Process for a Can***

Secret slides : Kubota case



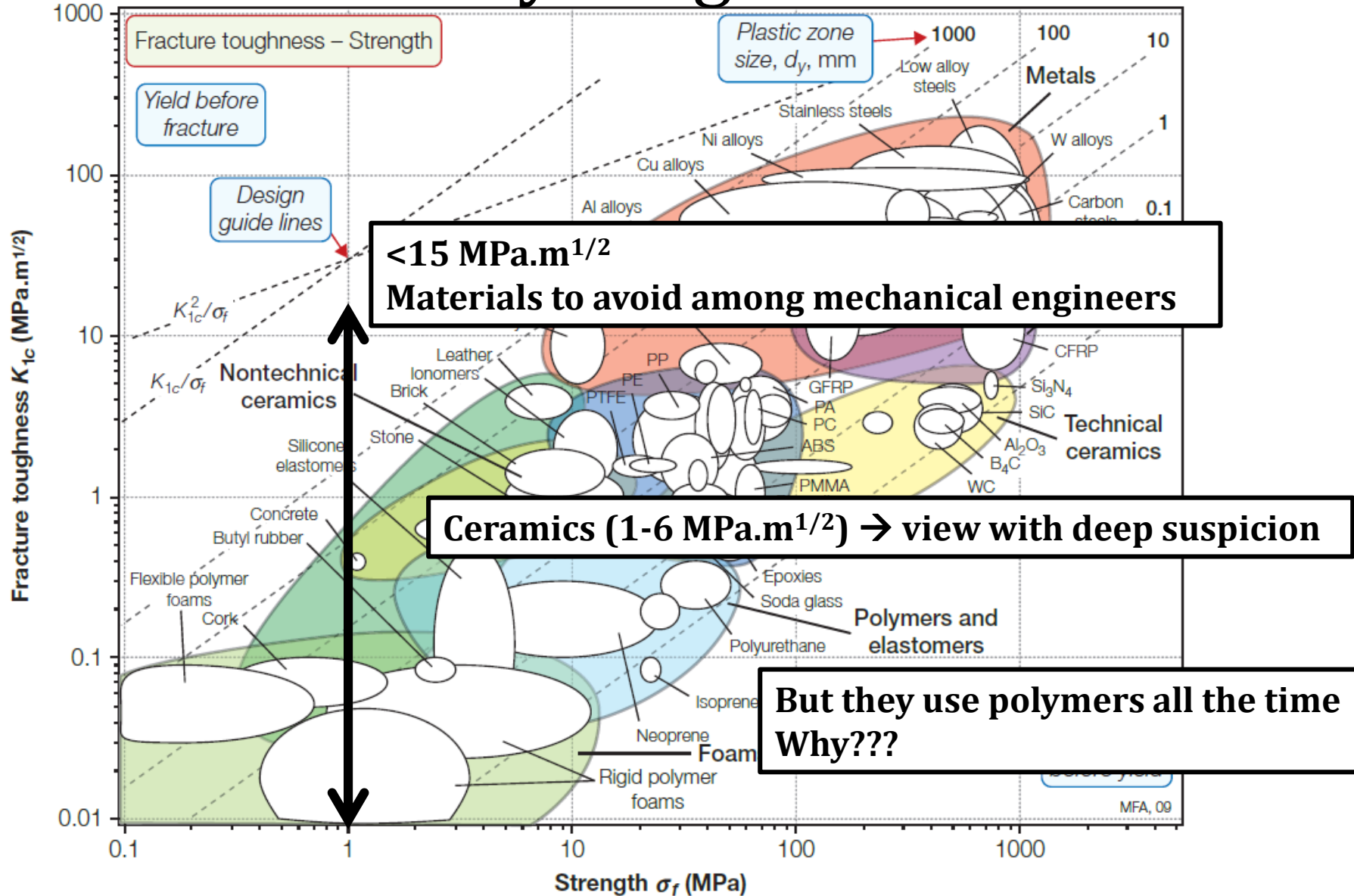
The Fracture-limited design



Fracture-limited design is important to control crack propagation



Ashby Diagrams





The Fracture-limited design

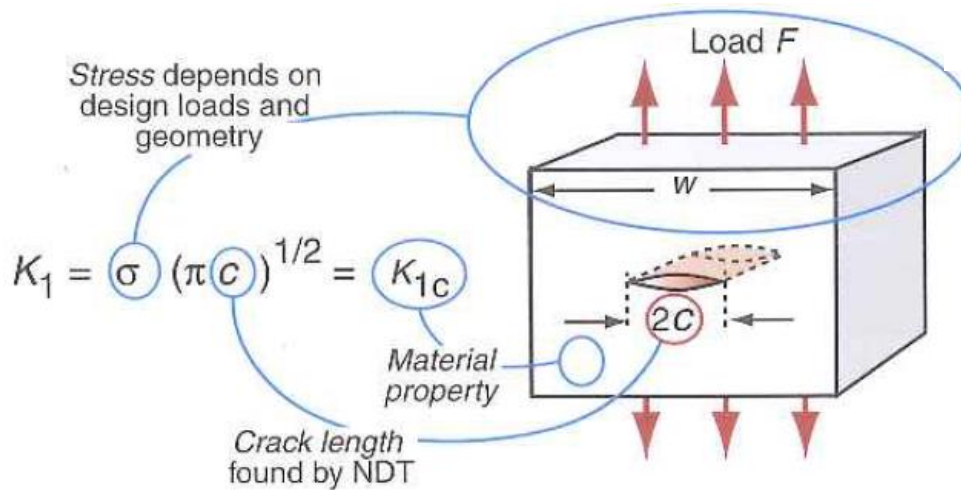
The resistance of a material to the propagation of a crack is measured by its plane-strain fracture toughness value (K_{1c})

When a brittle material is deformed, it deflects elastically until it fractures. The stress at which this happens is:

$$\sigma_f = \frac{K_{1c} \cdot C}{\sqrt{\pi \cdot a_c}}$$

σ_f by definition,
To consider all the stresses

K_1 = stress intensity factor
 $K_{1c} = K_1 \rightarrow$ crack propagates



NDT = non-destructive testing

Usually $C=1$

a_c = the length of the largest crack

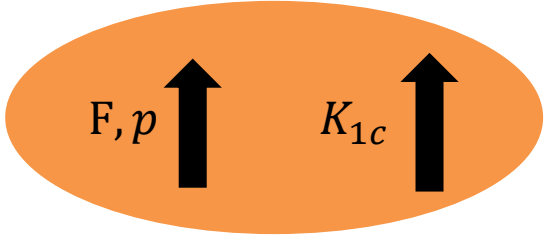


**Case Study 13:
Load-limited design**

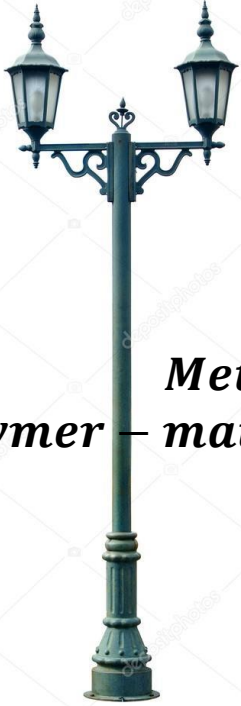
Objective	• Minimize the volume (mass, cost)
Constraints	• Design load specified
Free Variables	• Choice of material

*Carry a specified load
or pressure without fracturing?*

Easy

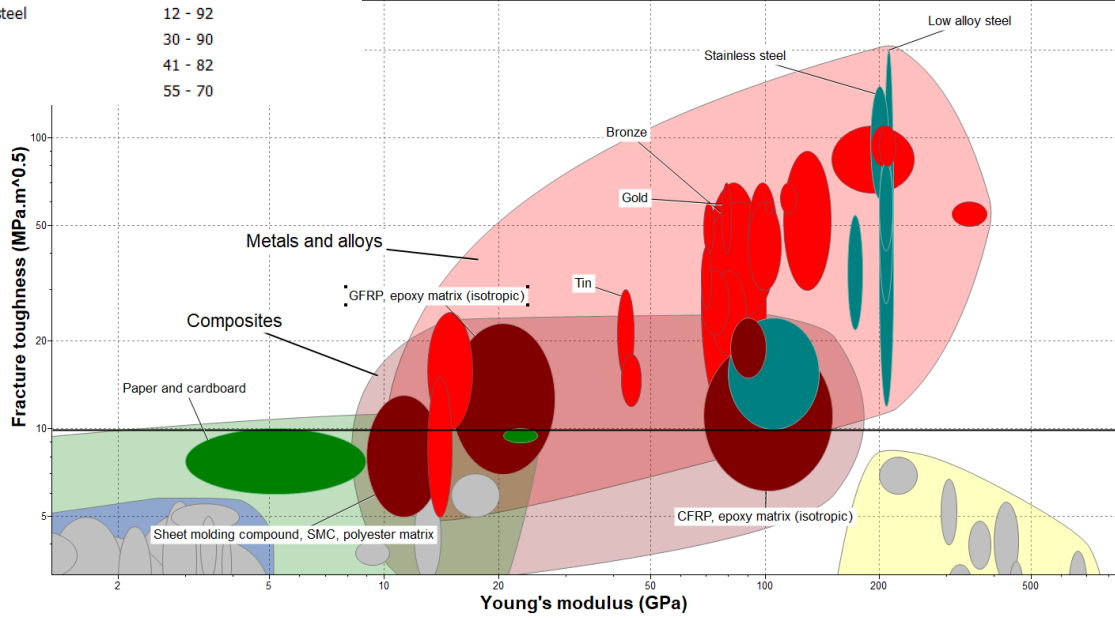


$$\sigma_f = \frac{K_{1c} \cdot C}{\sqrt{\pi \cdot a_c}}$$



**Metals,
polymer – matrix composites**

Name	Fracture toughness (MPa...)
Low alloy steel	14 - 200
Stainless steel	62 - 150
Nickel	80 - 110
Nickel-chromium alloys	80 - 110
Nickel-based superalloys	65 - 110
High carbon steel	27 - 92
Medium carbon steel	12 - 92
Copper	30 - 90
Low carbon steel	41 - 82
Titanium alloys	55 - 70





**Case Study 13:
Energy-limited design**

Objective	• Minimize the volume (mass, cost)
Constraints	• Design energy specified
Free Variables	• Choice of material

$$\left\{ \begin{array}{l} \sigma_f = \frac{K_{1c} \cdot C}{\sqrt{\pi \cdot a_c}} \\ W_{el} = \frac{\sigma_y^2}{2E} \end{array} \right.$$

Elastic energy



Springs

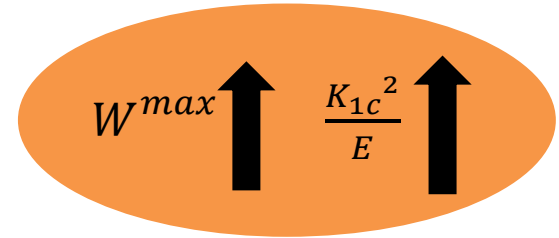


*Containment systems
for turbines*



Flywheels

$$W^{max} = \frac{C^2}{2\pi a_c} \cdot \frac{K_{1c}^2}{E}$$

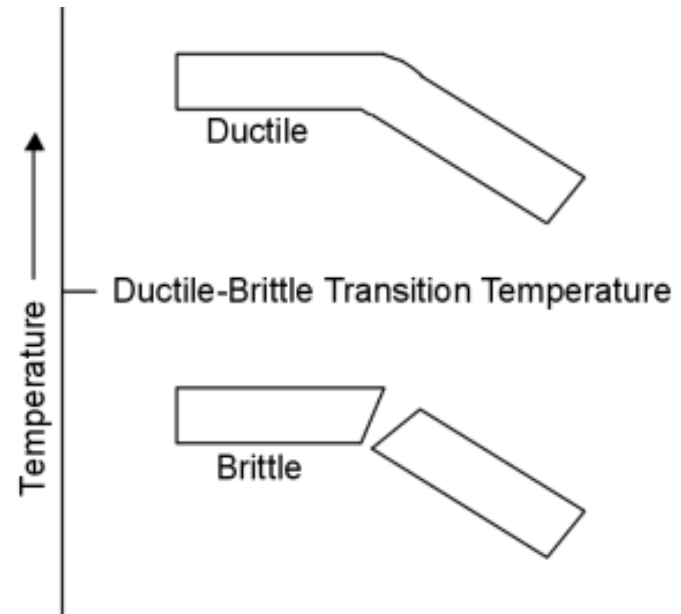
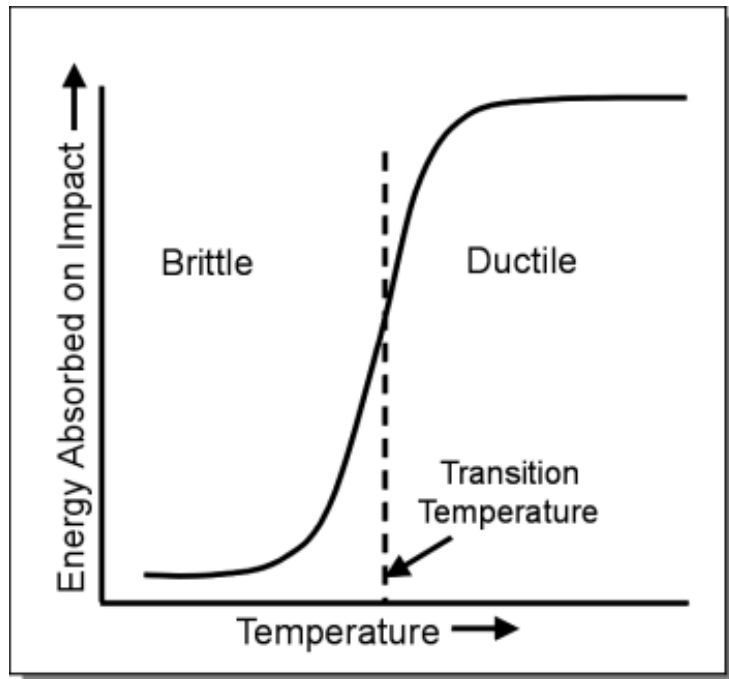


$$M_2 = \frac{K_{1c}^2}{E} \approx G_c$$

*Toughness
for a given flaw size*



Ductile-Brittle Temperature

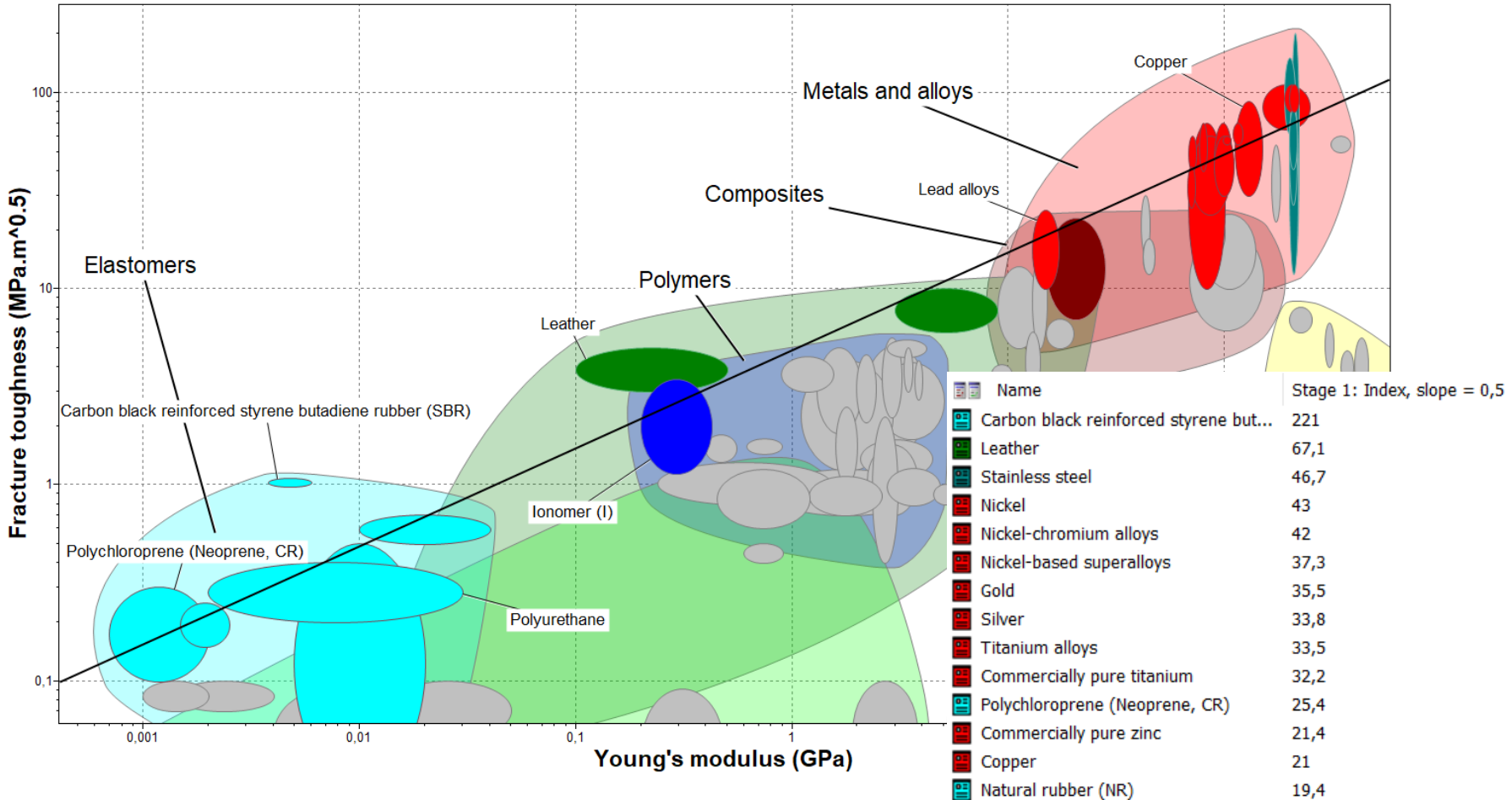




**Metals, composites
and some polymers**
 $J_c > 1 \text{ kJ/m}^2$

W^{max} ↑ $\frac{K_{1c}^2}{E}$ ↑

**Case Study 13:
Energy-limited design**





**Case Study 13:
Deflection-limited design**

Objective	• Minimize the volume (mass, cost)
Constraints	• Design deflection specified
Free Variables	• Choice of material

$$\left\{ \begin{array}{l} \sigma_f = \frac{K_{1c} \cdot C}{\sqrt{\pi \cdot a_c}} \\ W_{el} = \frac{\sigma_y^2}{2E} \\ \varepsilon = \frac{\sigma}{E} \end{array} \right.$$



Snap on bottle tops

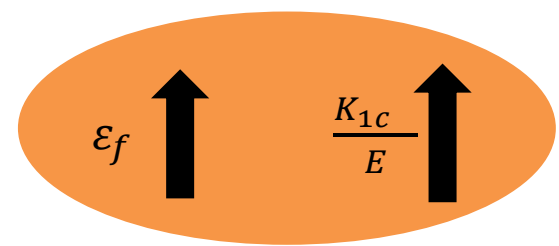


Snap-together fasteners

We want a large failure strain

$$\varepsilon_f = \frac{C}{\sqrt{\pi a_c}} \cdot \frac{K_{1c}}{E}$$

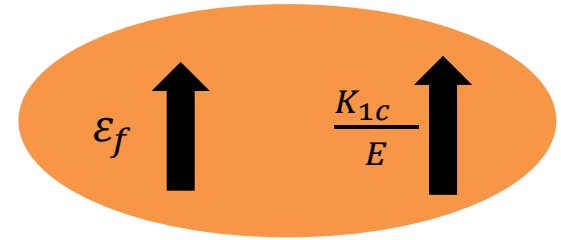
$$M_2 = \frac{K_{1c}}{E}$$



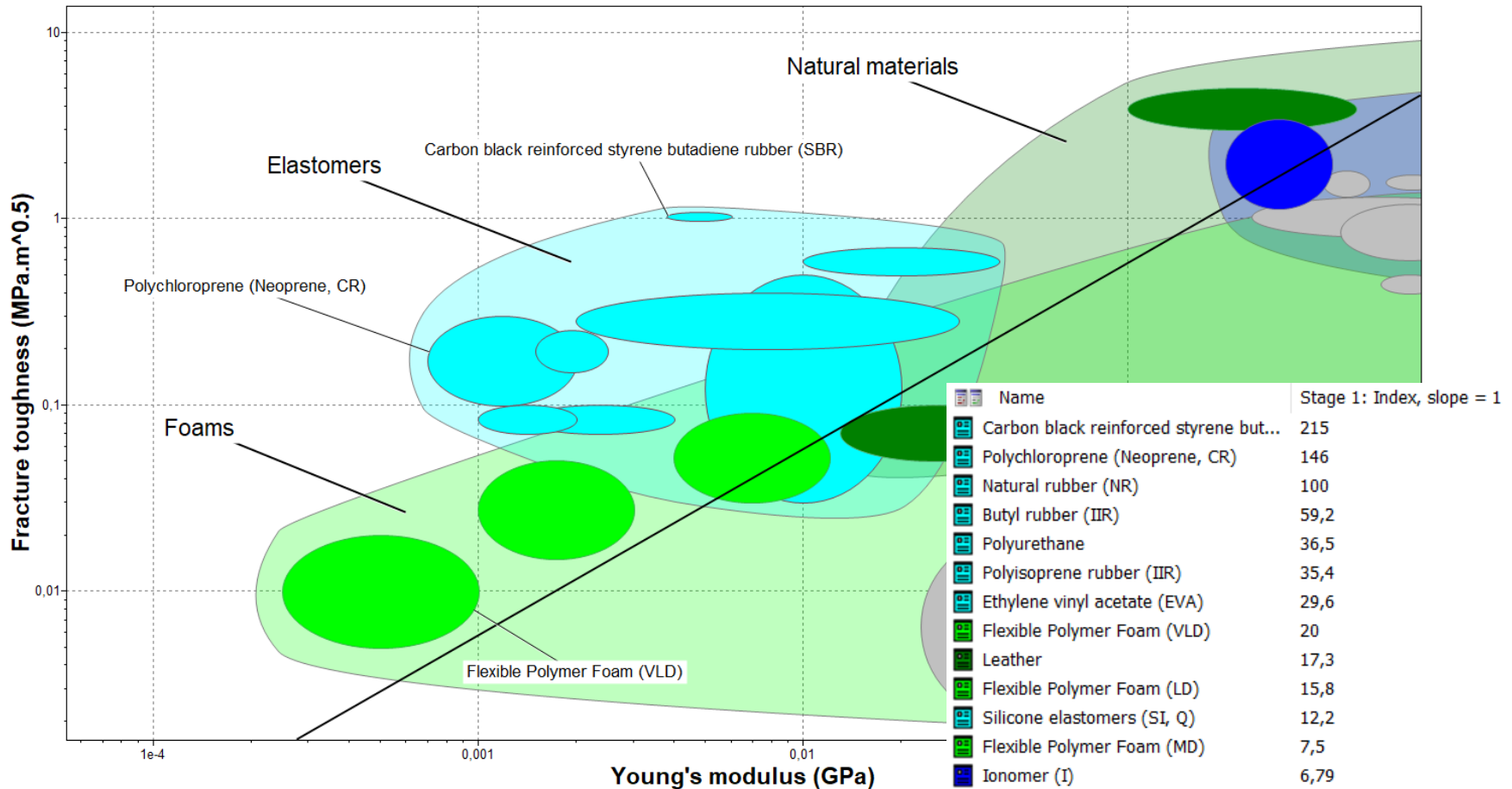


Polymers, elastomers and toughest metals

$$\frac{K_{1c}}{E} > 10^{-3} \text{ m}^{1/2}$$

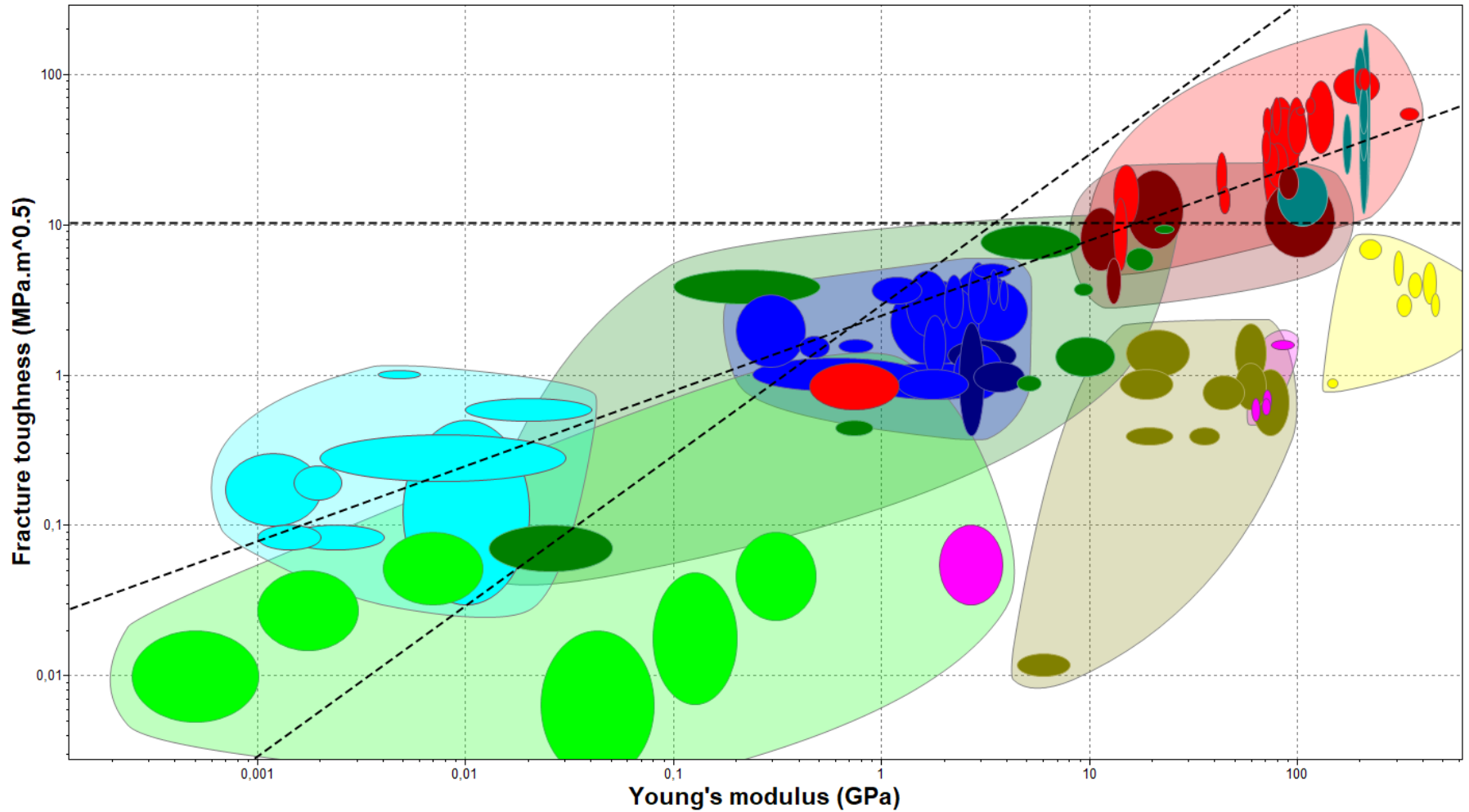


Case Study 13: Deflection-limited design





The Fracture-limited design



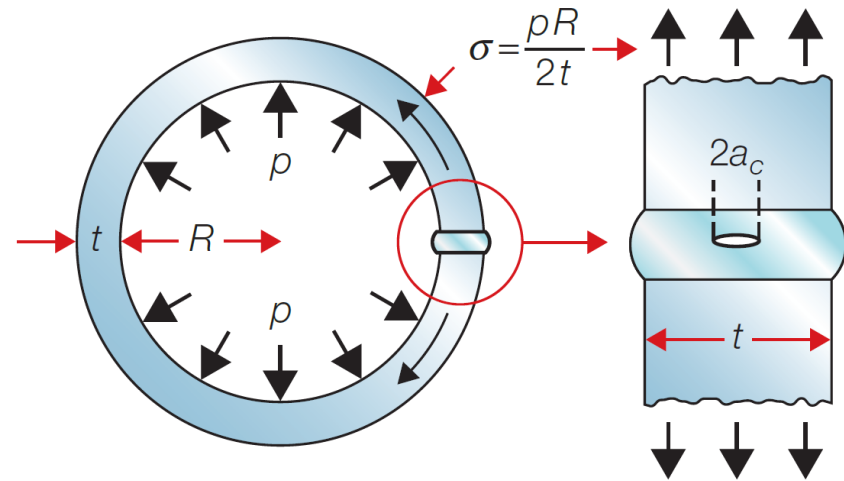


Case Study 14: Safe Pressure Vessels

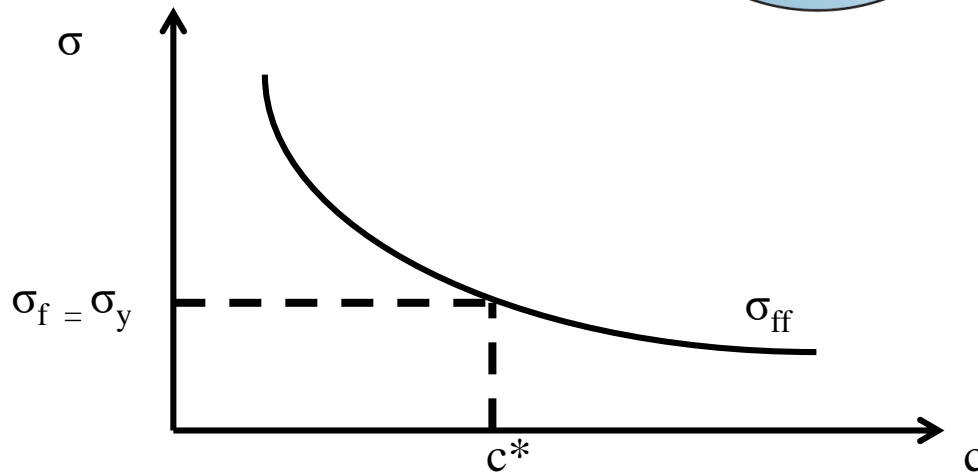




Case Study 14: Safe Pressure Vessels



Two different approaches



Yield-Before-Break (YBB)

Leak-Before-Break (LBB)

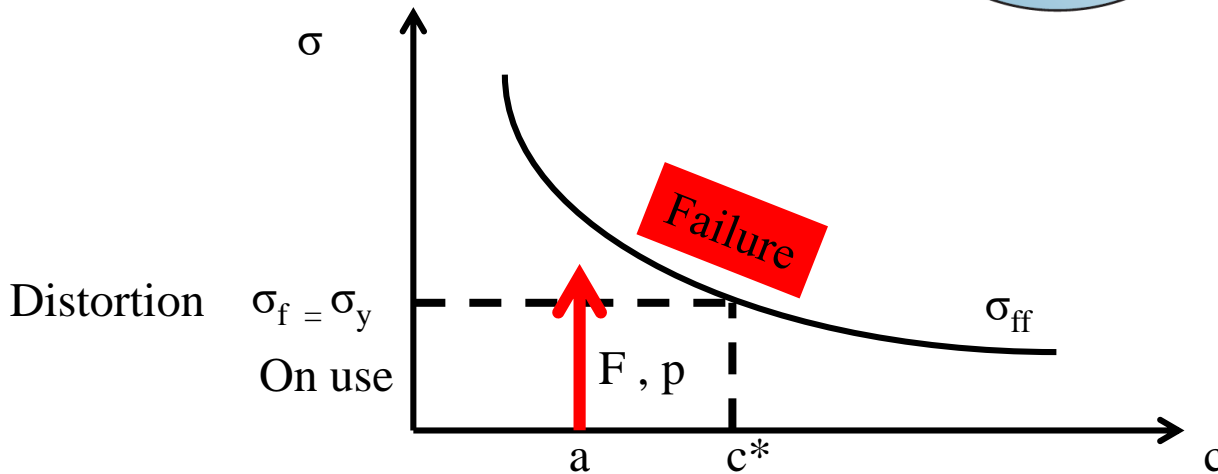
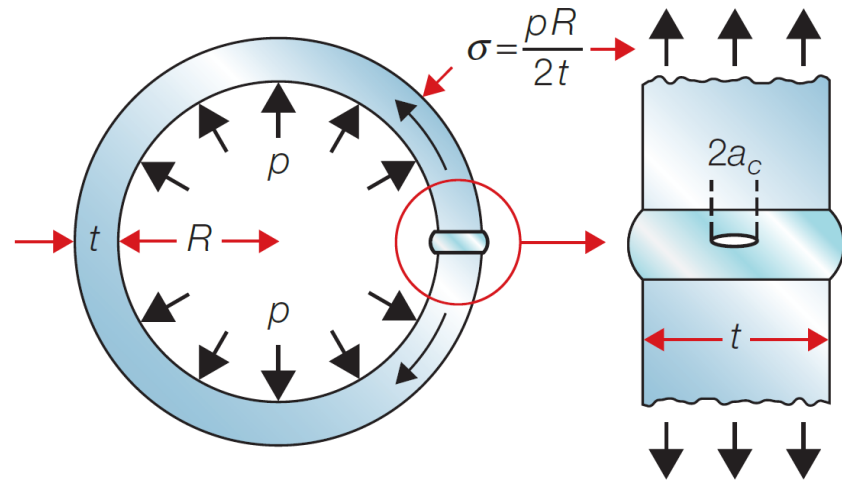
No failure, but distortion
Small vessels are designed
to allow general yield

Safe design
Smallest crack has a thickness greater
than the thickness of the vessel wall



**Case Study 14:
Safe Pressure Vessels
YBB**

Two different approaches



Yield-Before-Break (YBB)

No failure, but distortion
Small vessels are designed
to allow general yield

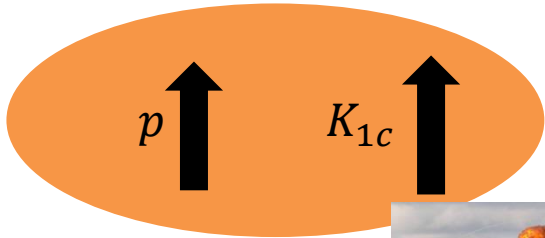


**Case Study 14:
Safe Pressure Vessels
YBB**

Objective	• Maximize safety (YBB)
Constraints	• R radius specified
Free Variables	• Choice of material

$$\left\{ \begin{array}{l} \sigma \leq \frac{C \cdot K_{1c}}{\sqrt{\pi \cdot a_c^*}} \\ \sigma = \frac{p \cdot R}{2t} \end{array} \right.$$

$$p \leq \frac{2t \cdot K_{1c}}{R \cdot \sqrt{\pi \cdot a_c^*}}$$



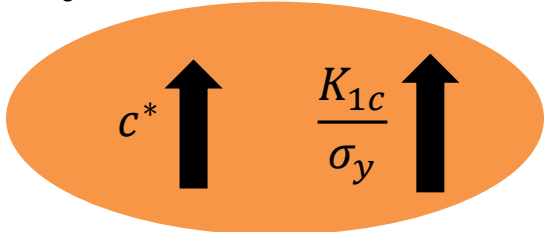
**But not
Fail-safe**

If the inspection is faulty for some reasons the crack length is greater



$$\pi \cdot 2 \cdot c^* \leq C^2 \cdot \left(\frac{K_{1c}}{\sigma_y} \right)^2$$

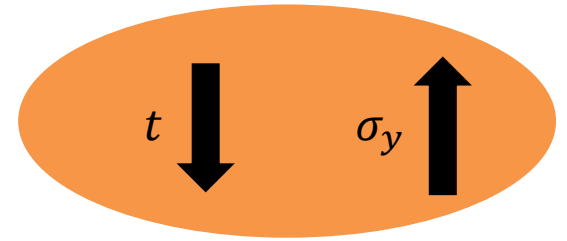
a_c Tolerable crack size



$$M_1 = \frac{K_{1c}}{\sigma_y}$$

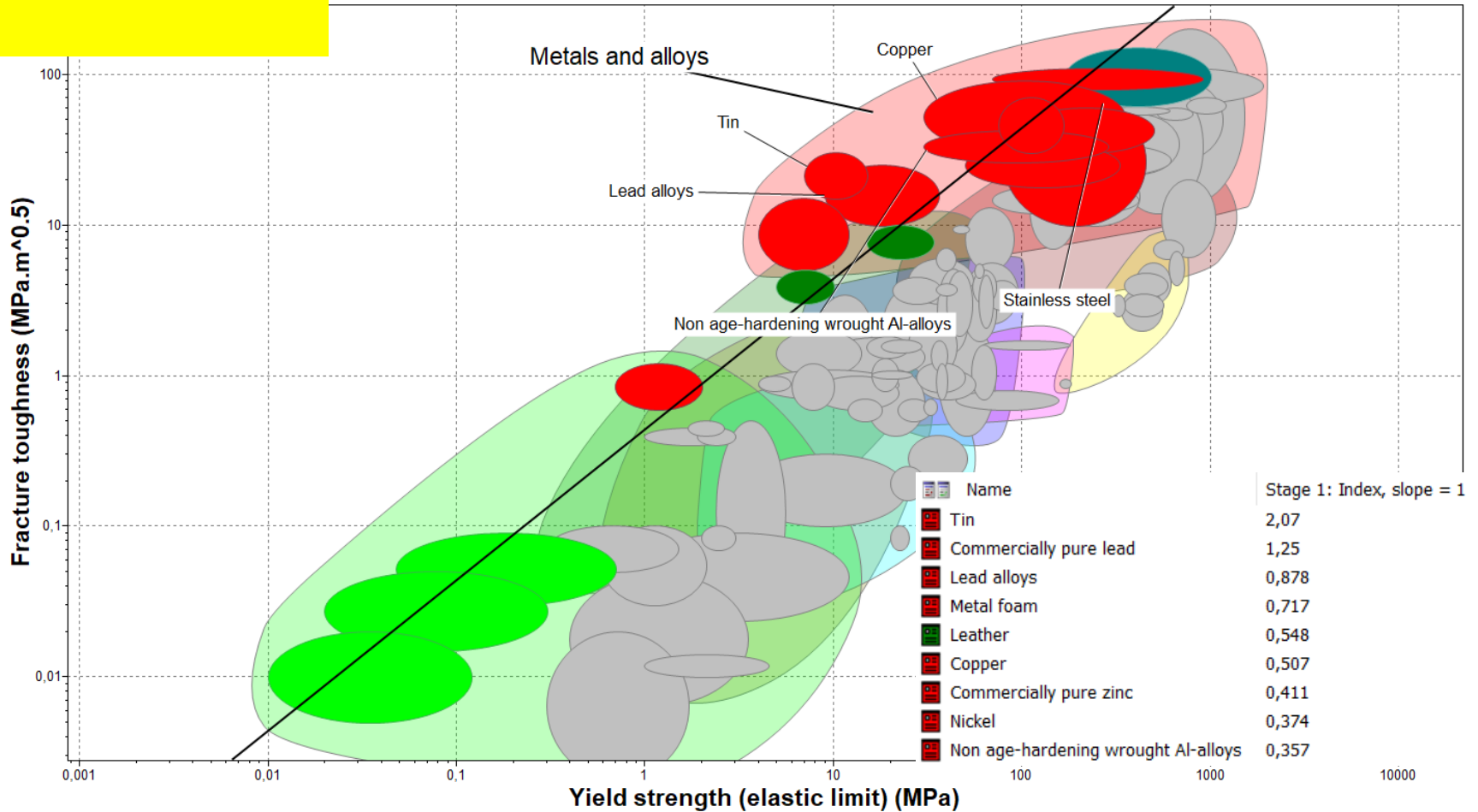


Since $t = \frac{p \cdot R}{2\sigma}$



For Thinner t

**Case Study 14:
Safe Pressure Vessels
YBB**

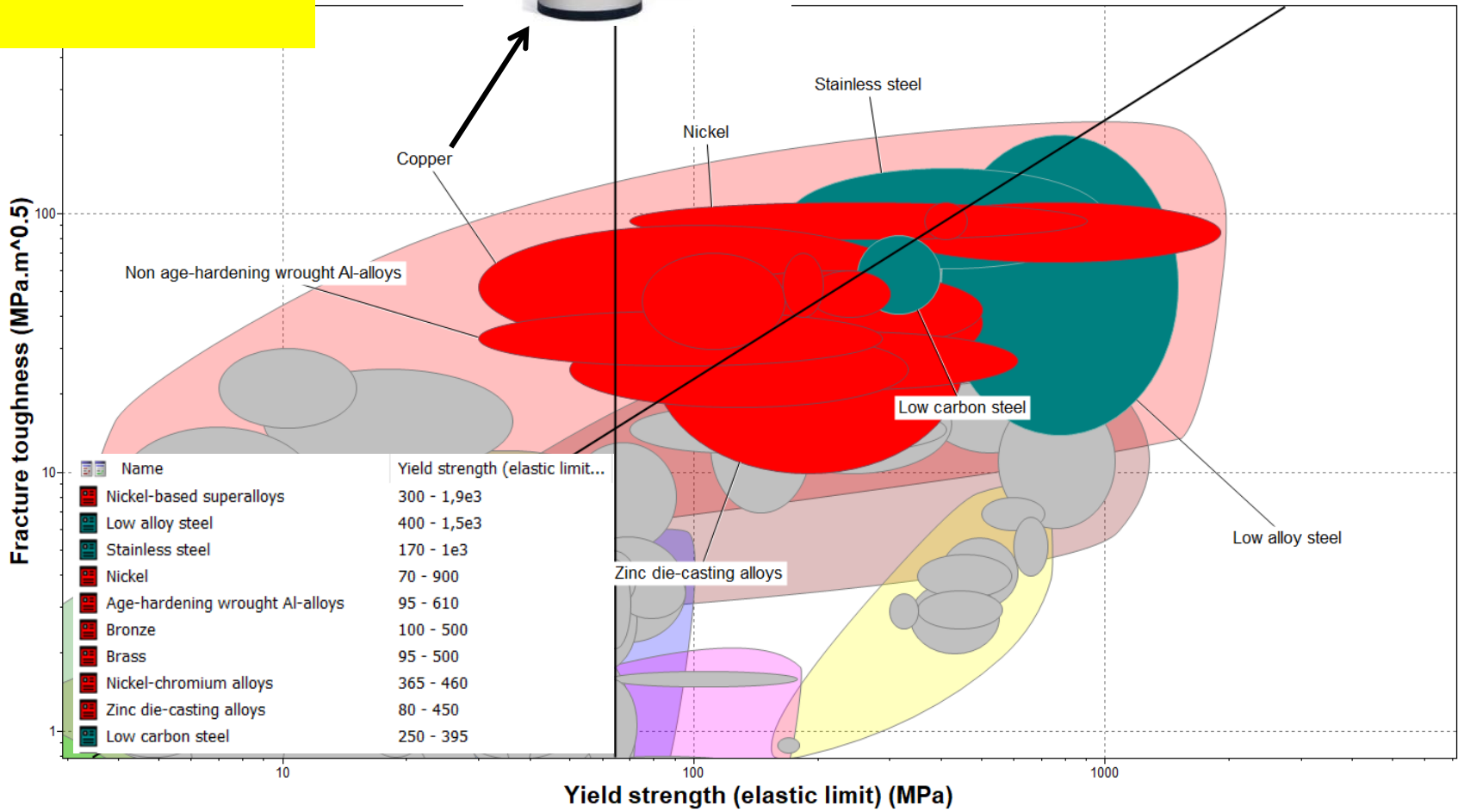




*Small boilers
(Hard drawn copper)*

Low alloys steel = Classical choice

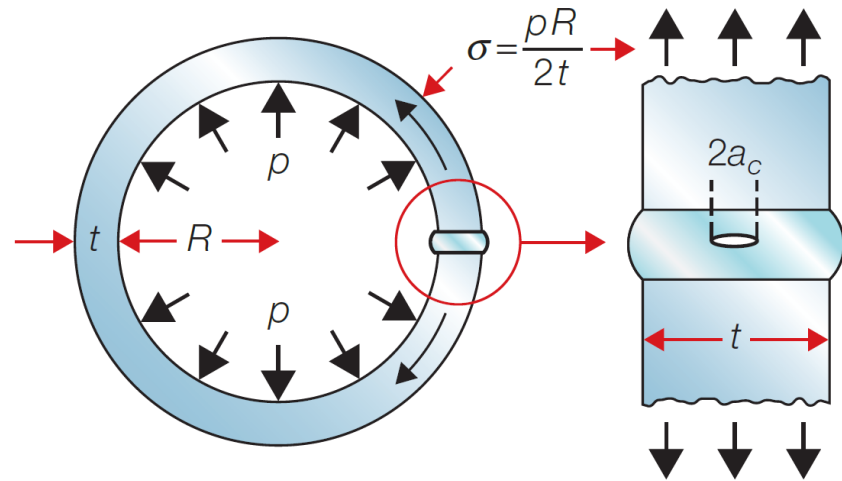
**Case Study 14:
Safe Pressure Vessels
YBB**



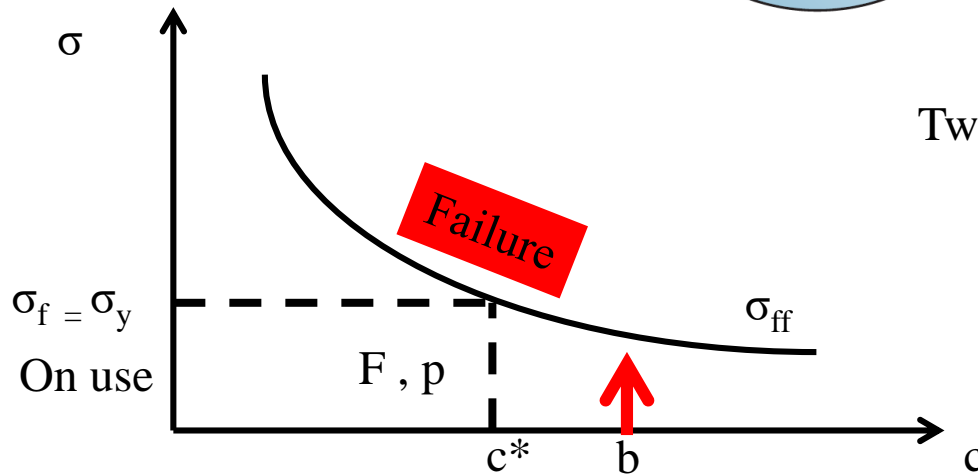


**Case Study 14:
Safe Pressure Vessels
LBB**

Two different approaches



Two different approaches



Leak-Before-Break(LBB)

Safe design

Smallest crack has a thickness greater than the thickness of the vessel wall



**Case Study 14:
Safe Pressure Vessels
LBB**

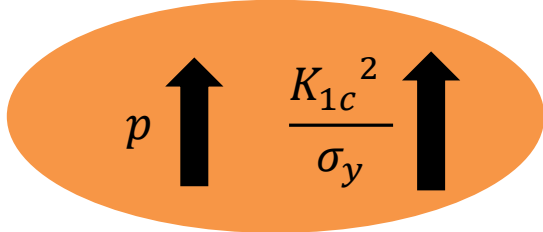
Objective	• Maximize safety (LBB)
Constraints	• R radius specified
Free Variables	• Choice of material

Hp: $b = \frac{t}{2}$

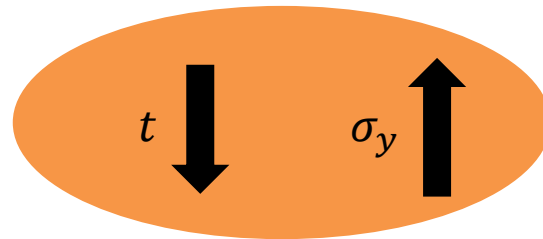
$$\left\{ \begin{array}{l} \sigma \leq \frac{C \cdot K_{1c}}{\sqrt{\pi \cdot t/2}} \\ \sigma_y = \frac{p \cdot R}{2t} \end{array} \right. \rightarrow t = \frac{p \cdot R}{2\sigma_y}$$

$$p \leq \frac{4C^2}{\pi R} \cdot \left(\frac{K_{1c}^2}{\sigma_y} \right) \quad M_2 = \frac{K_{1c}^2}{\sigma_y}$$

a_c **Tolerable crack size**



$$t = \frac{p \cdot R}{2\sigma_y} \quad \text{Thinner } t$$





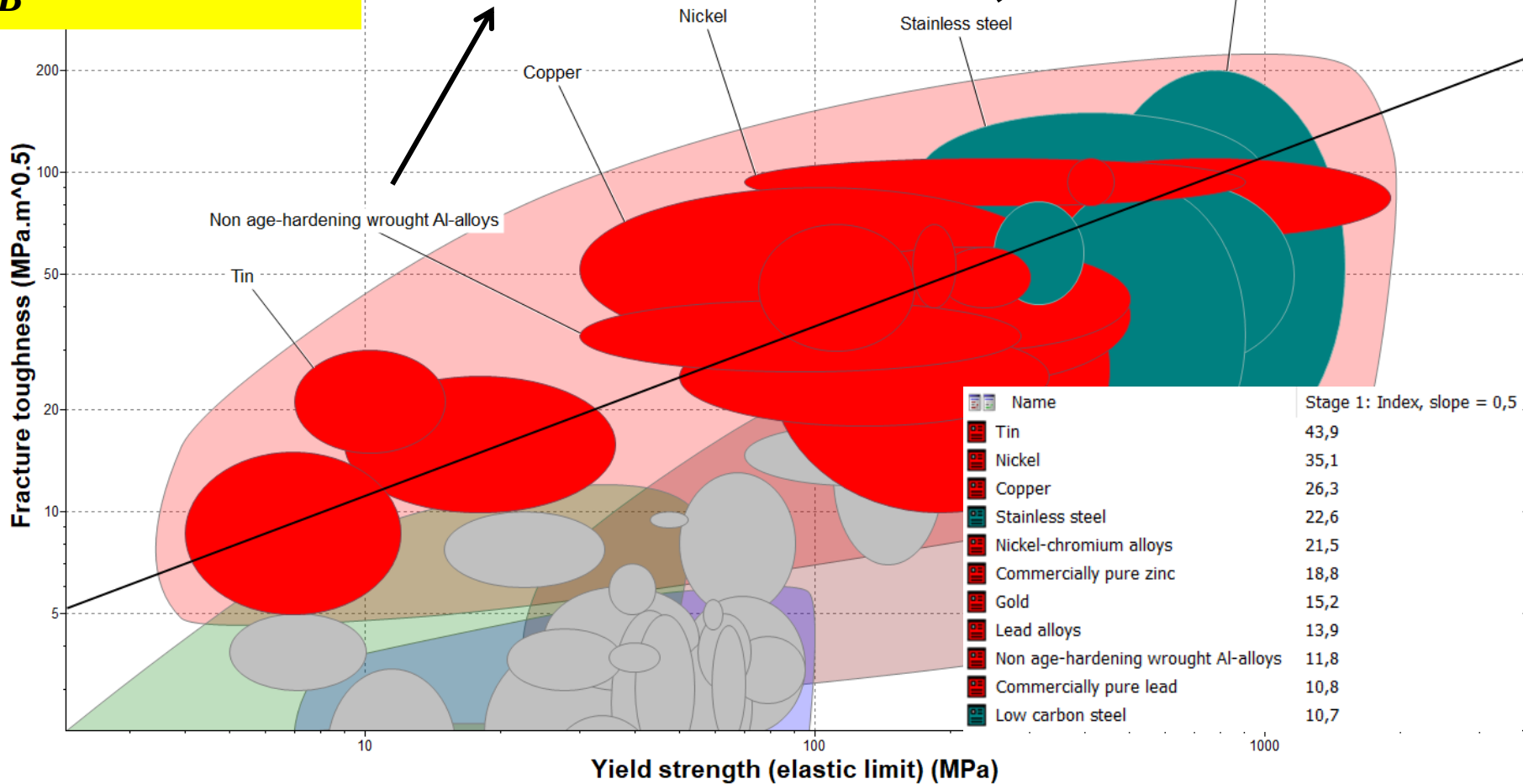
Pressure tanks of rockets



Nuclear pressure vessels (316 Stainless Steel)

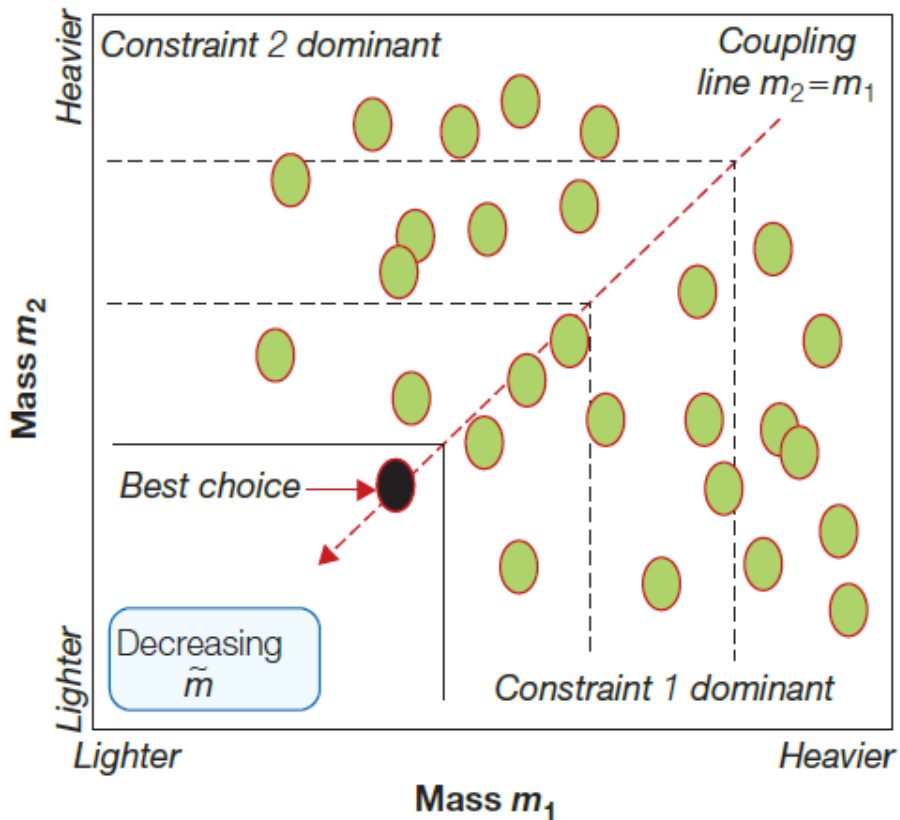


**Case Study 14:
Safe Pressure Vessels
LBB**

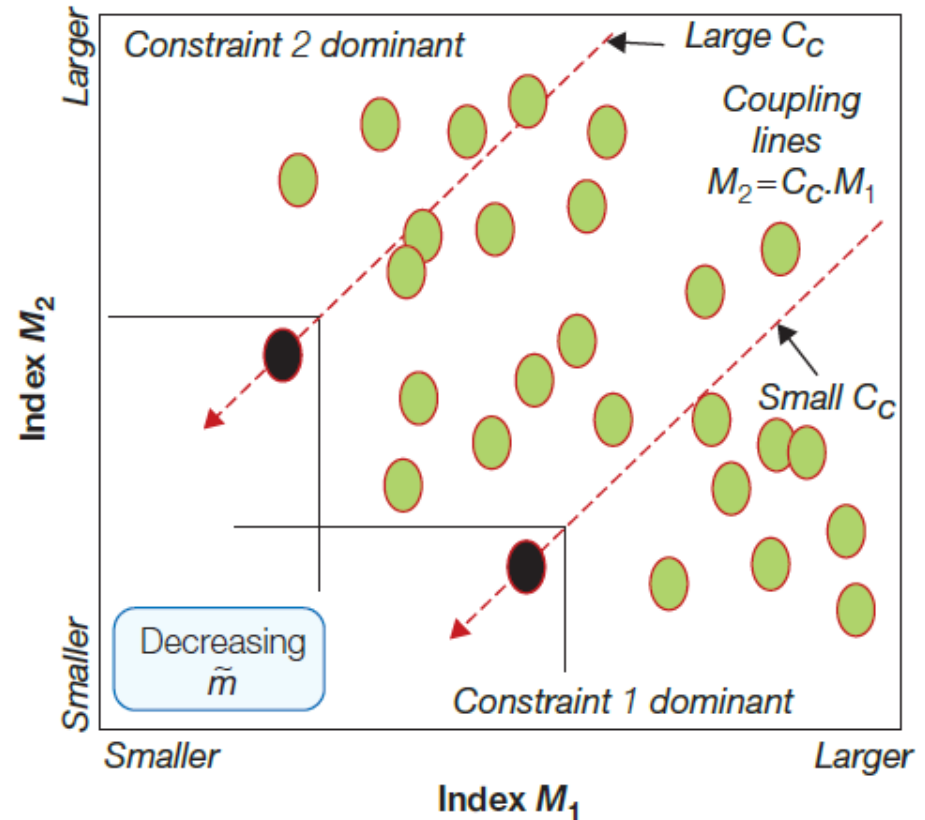




Multiple constraints



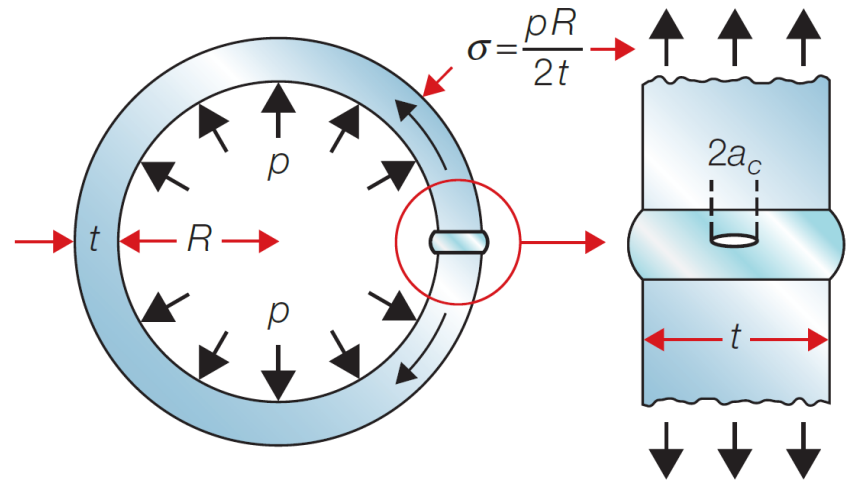
Minimize the mass



Minimize the volume



**Case Study 14':
Materials for Light pressure Vessels
With MULTIPLE CONSTRAINT
APPROACH**



Fracture constraint

$$\sigma = \frac{p \cdot R}{2t} \leq \frac{K_{1c}}{\sqrt{\pi \cdot c}}$$

$$m = 4\pi R^2 \cdot t \cdot \rho$$

Yield constraint

$$\sigma = \frac{p \cdot R}{2t} \leq \sigma_y$$

$$m_1 = 2\pi R^3 \cdot p \cdot \sqrt{\pi \cdot c} \cdot \frac{\rho}{K_{1c}}$$

$$M_1 = \frac{\rho}{K_{1c}}$$

$$m_2 = 2\pi R^3 \cdot p \cdot \frac{\rho}{\sigma_y}$$

$$M_2 = \frac{\rho}{\sigma_y}$$

Objective

Constraints

Performance equation

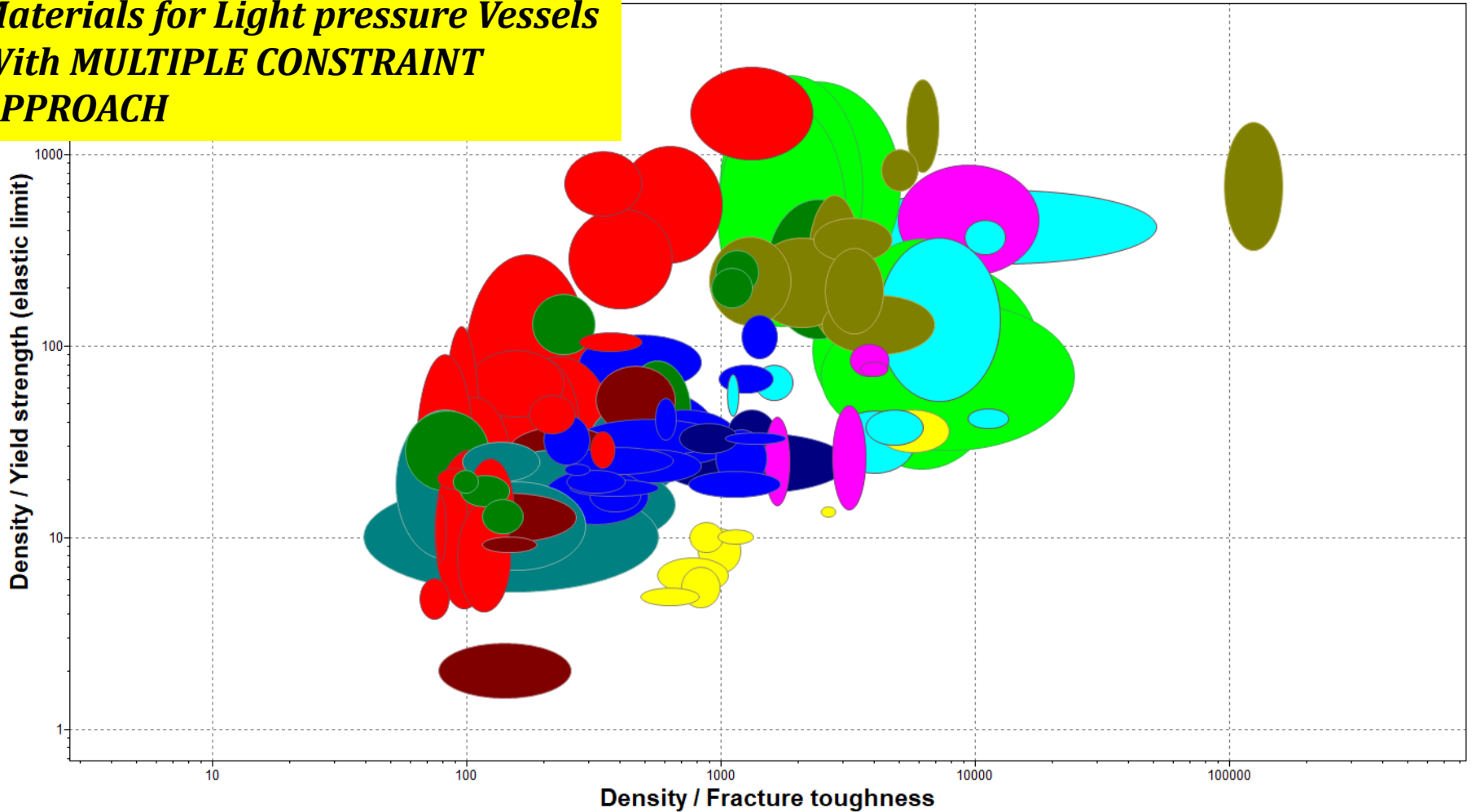
Index



$$\frac{\rho}{\sigma_y} = \sqrt{\pi \cdot c} \cdot \frac{\rho}{K_{1c}}$$

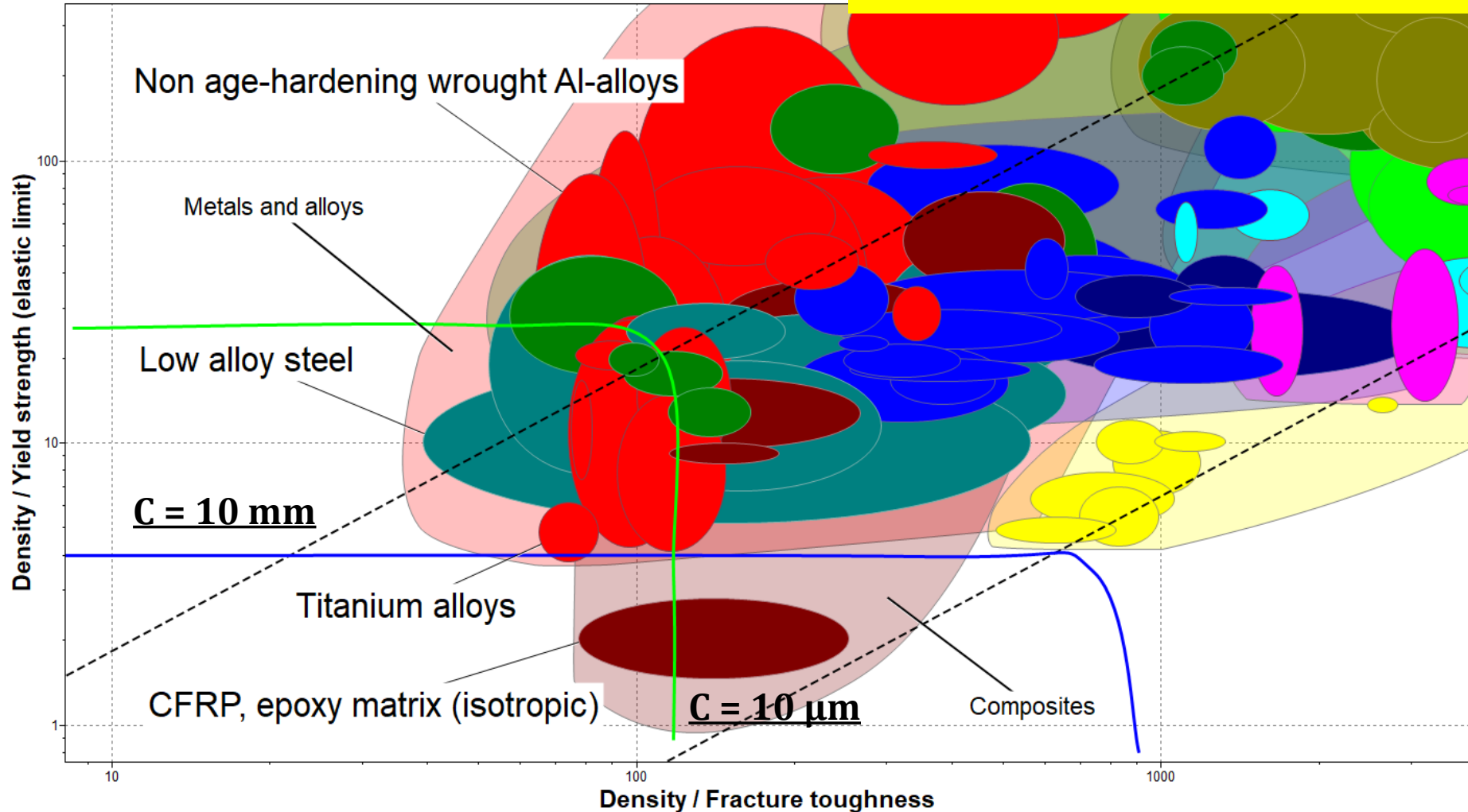
$$M_2 = \sqrt{\pi \cdot c} \cdot M_1$$

**Case Study 14':
Materials for Light pressure Vessels
With MULTIPLE CONSTRAINT
APPROACH**





**Case Study 14':
Materials for Light pressure Vessels
With MULTIPLE CONSTRAINT
APPROACH**





Theory: Method of the weight factor

$$Z_p = \alpha_H \cdot \frac{H}{\max(H)} + \alpha_M \cdot \frac{M}{\max(M)}$$

The most important point is to understand if you must maximize or minimize a property

Example:

Young's Modulus *Density*

Maximize *E* $1/\rho$

Minimize $1/E$ ρ

With the normalization and the performance function you can consider all the properties that you want and with various order of magnitude.

But the importance coefficients? → **EDL**

Example of its application

7.6. EDL method and Performance function

The performance function is a selection method to analyze different materials and to discover which is the most proper for a particular utilization; the function can correlate many different material properties and based on the requirements it is possible assign an importance sequence with the EDL (Enhanced Digital Logic). In the study this methods were utilized to correlate the hardness values and the Magneqage values. The first is a complex value to understand because is due to the matrix and to the carbide present, instead the second is a value difficult to correlate due to the precise surveys and to the empiric bases of the analyses. In a HSS a property that is consider fundamental is the low percentage of retained austenite, thus the function is based on this assumption; as importance sign (α) the EDL method assigns 0.33 for the hardness and 0.67 for the retained austenite. The performance function is:

$$Z_p = \alpha_H \cdot \frac{H}{\max(H)} + \alpha_M \cdot \frac{M}{\max(H)} \quad (5)$$

Equation 5: Performance function of the thermal treatments.

H and M are the sample values and they are divided for the maximum between them; with this important step the values are normalized, as consequence they can be correlated with other different properties and properties as Magneqage analysis can have a higher import. In the Table 33 all the performance function results (Z_p) are reported in the two relevant depths; to arrive at those values the data from Table 29 and Table 30 were used.

	R1	N0	N1	N2	N3	N4	N5	X1/X3	N6
20	0,25	0,44	0,75	0,97	0,93	0,91	0,97	0,89	0,80
50	0,54	0,47	0,68	0,80	0,78	0,80	0,91	0,92	0,77

Table 33: Results of the performance function.

The samples are reported in order from the R1 (different thermal treatment) and N0 (as-cast conditions) to N6 that has the higher temperature of thermal treatment (double tempering). The results are putted in order from the higher value to the lower one in the two depths as shown in the Table 34.

Sample	20	Sample	50	Sample	Mean value
N2 (450)	0,97	X1/X3 (530)	0,92	N5(525)	0,94
N5(525)	0,97	N5(525)	0,91	X1/X3 (530)	0,905
N3 (475)	0,93	N2 (450)	0,8	N2 (450)	0,885
N4 (500)	0,91	N4 (500)	0,8	N3 (475)	0,855
X1/X3 (530)	0,89	N3 (475)	0,78	N4 (500)	0,855
N6 (550)	0,8	N6 (550)	0,77	N6 (550)	0,785
N1 (425)	0,75	N1 (425)	0,68	N1 (425)	0,715
N0 (44)	0,44	R1 (0,54)	0,54	N0	0,455
R1	0,25	N0	0,47	R1	0,395

Table 34: The results of Table 33 in order from the highest value to the lowest for the two depths (A and B). The different samples are written together with the thermal treatment temperature. The mean values of the two depths for every sample in (C).

The best material and in this case the best thermal treatment that can be applied on the material is the double tempering in the range of temperature 525-530°C as reported in the following table; to arrive at this conclusion it was done the mean value of the two depth values to try to consider the entire roll and not only a single part of it.

on a process



Theory: Enhanced Digital Logic

$$Z_p = \alpha_H \cdot \frac{H}{\max(H)} + \alpha_M \cdot \frac{M}{\max(M)}$$

σ Yield Strength
W Weldability

C Corrosion Resistance
F Fatigue Resistance

A really more important than B

3-1

A more important than B

2-1

A as imp. as B

1-1

Importance Sequence

$\sigma > W = C > F$

							Addition		
σ	2	2	3				7	0,389	0,39
W	1			1	2		4	0,222	0,22
C		1		1		2	4	0,222	0,22
F			1		1	1	3	0,167	0,17
							18		1



Theory: Enhanced Digital Logic

And if I want to have a property value similar to another material?

→ Normalisation MANCINI-MAURIZI

Example : Coefficient of thermal expansion α (CTE)

If the property is bigger than the target value
$$N_n = 1 - \frac{CTE_n - CTE_{targ.}}{CTE_{max} - CTE_{targ.}}$$

If the property is smaller than the target value
$$N_n = 1 - \frac{CTE_{targ.} - CTE_n}{CTE_{targ.} - CTE_{min}}$$

Again, a value close to 1 = the two CTEs are similar (the contrary to 0)



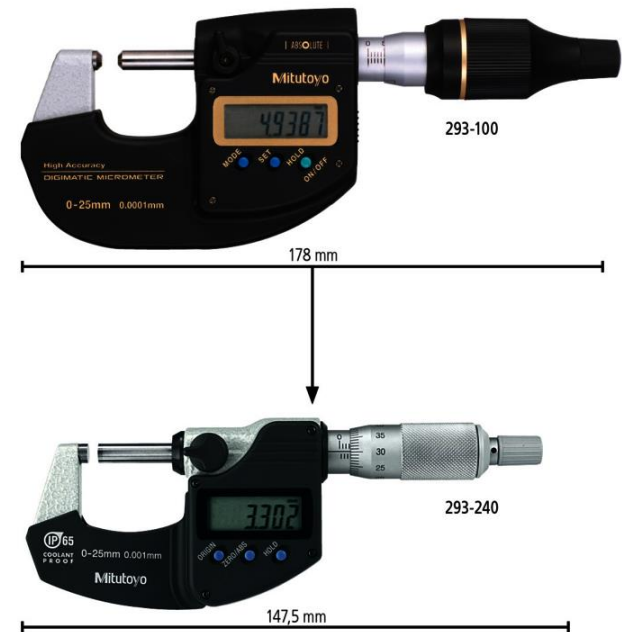
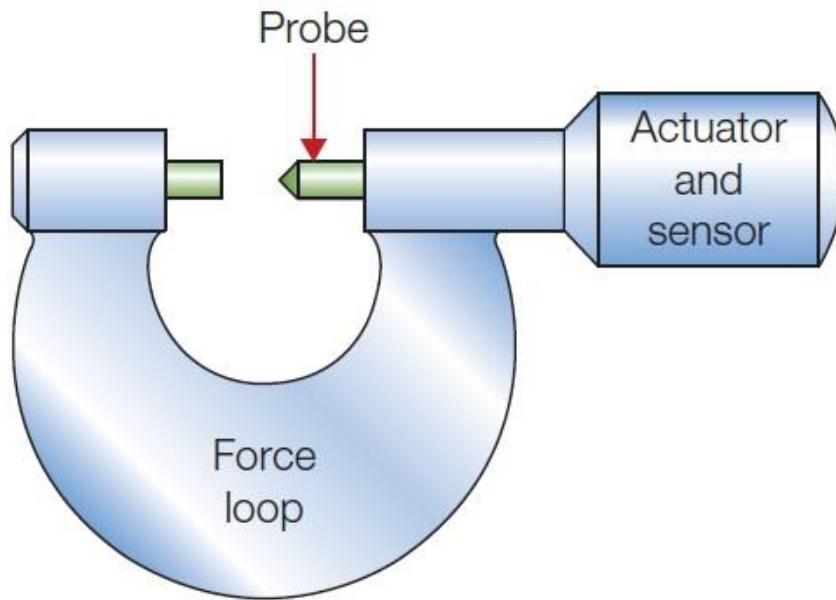
Using materials at high temperatures





Case Study 15:
Materials for Precision devices

Objective	<ul style="list-style-type: none">Minimize distortion to maximize positional accuracy
Constraints	<ul style="list-style-type: none">Tolerate heat fluxTolerate vibration
Free Variables	<ul style="list-style-type: none">Choice of material



Es. Sub micrometer displacement gauge



**Case Study 15:
Materials for Precision devices**

Fourier's Law in steady state

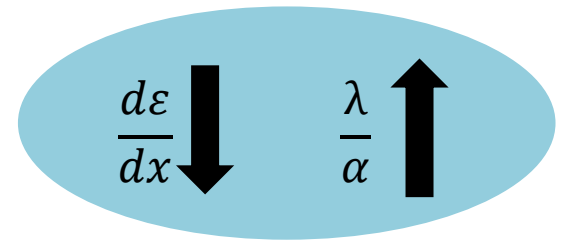
$$q = -\lambda \frac{dT}{dx}$$

$$\varepsilon = \alpha(T - T_0)$$

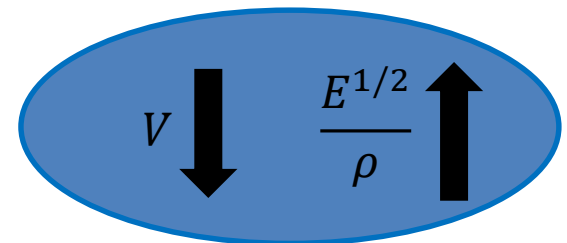
q Heat flux
 λ Thermal conductivity
 $\frac{dT}{dx}$ resulting temperature gradient

The distortion is proportional to the gradient of strain

$$\frac{d\varepsilon}{dx} = \alpha \cdot \frac{dT}{dx} = \frac{\alpha}{\lambda} \cdot q$$



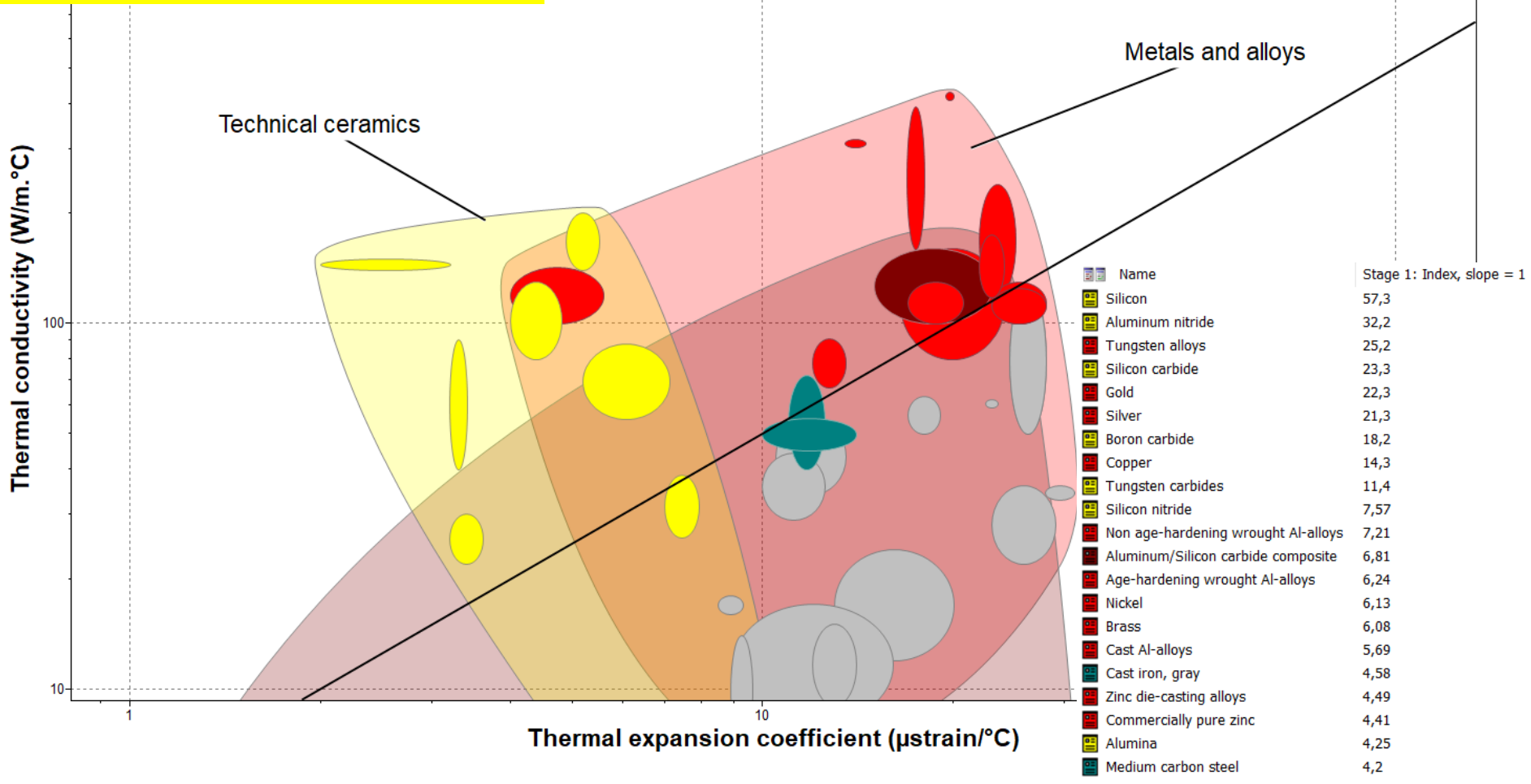
Moreover, we want to avoid flexural vibrations at lowest frequencies; proportional to (finally must be stiff) →





$$\frac{d\varepsilon}{dx} \downarrow \quad \lambda \uparrow \quad \frac{\lambda}{\alpha}$$

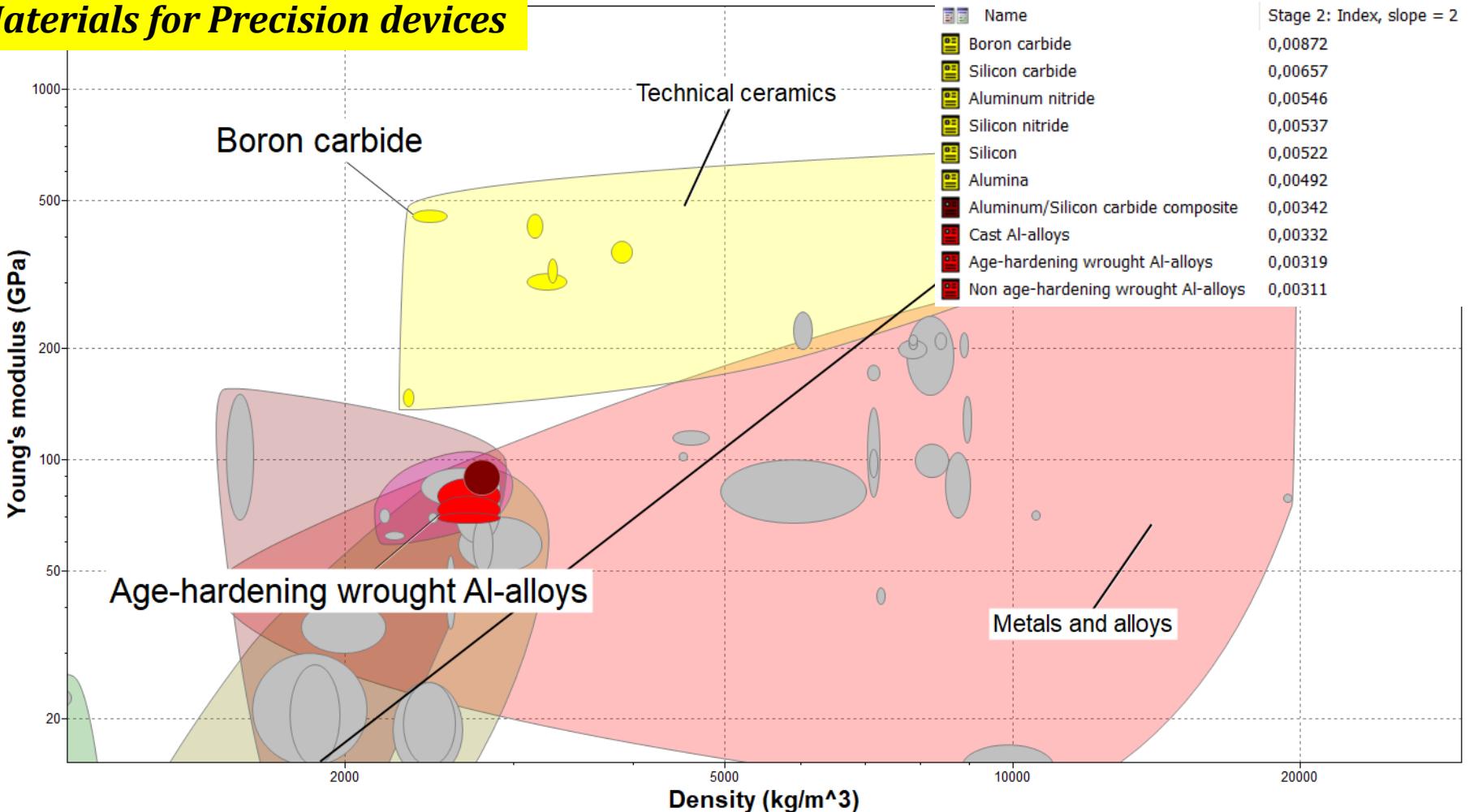
Case Study 15: Materials for Precision devices





Carbides? → Excellent performances, but difficult to shape
Copper → High density gives a low M2
Al alloys → the cheapest and most easily shaped choice
Silicon → the best choice

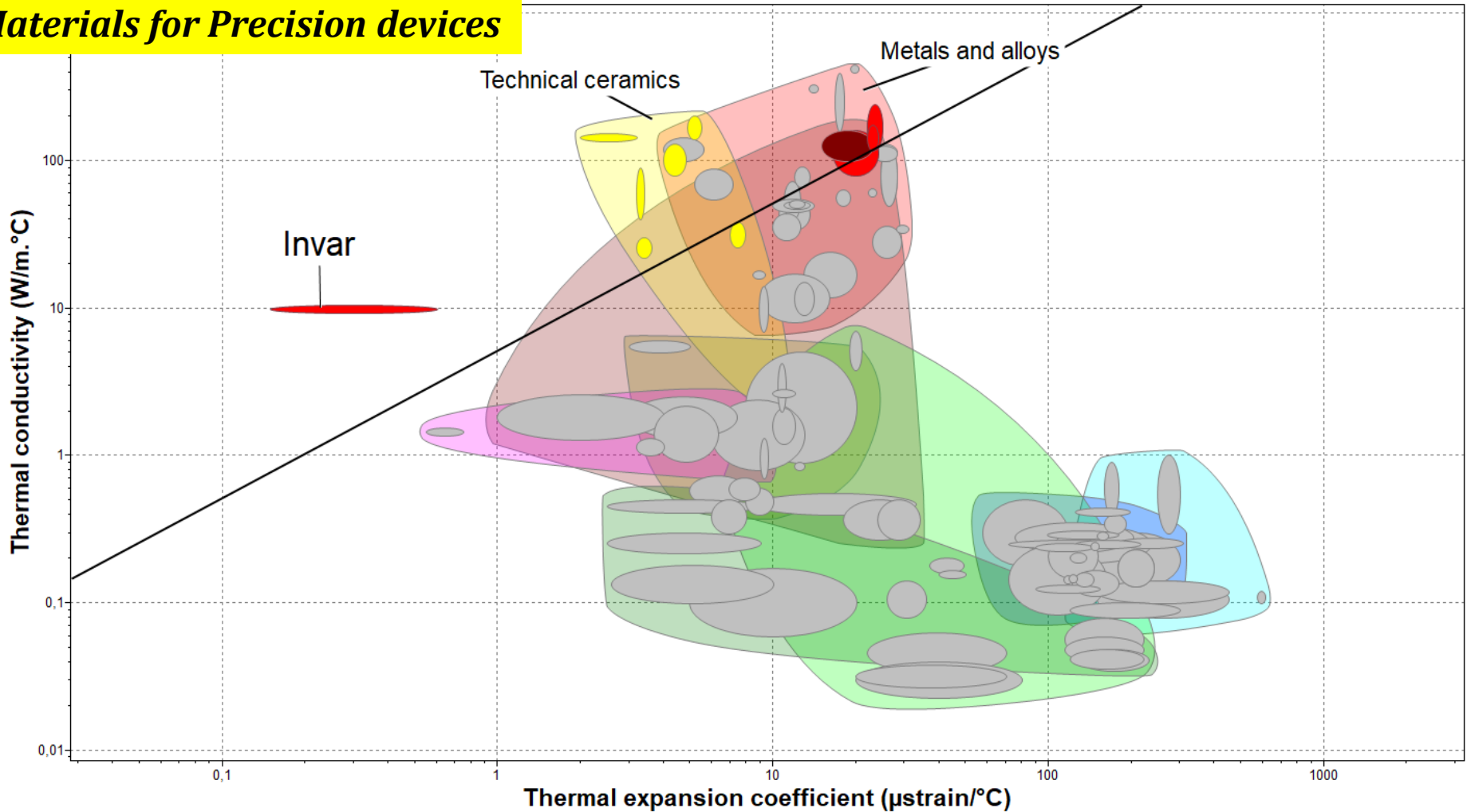
Case Study 15: Materials for Precision devices





With some researches you can find better materials!
Ex. Invar
Add Record IF IT HAS A SENSE

Case Study 15:
Materials for Precision devices





Conductors, insulators and dielectrics

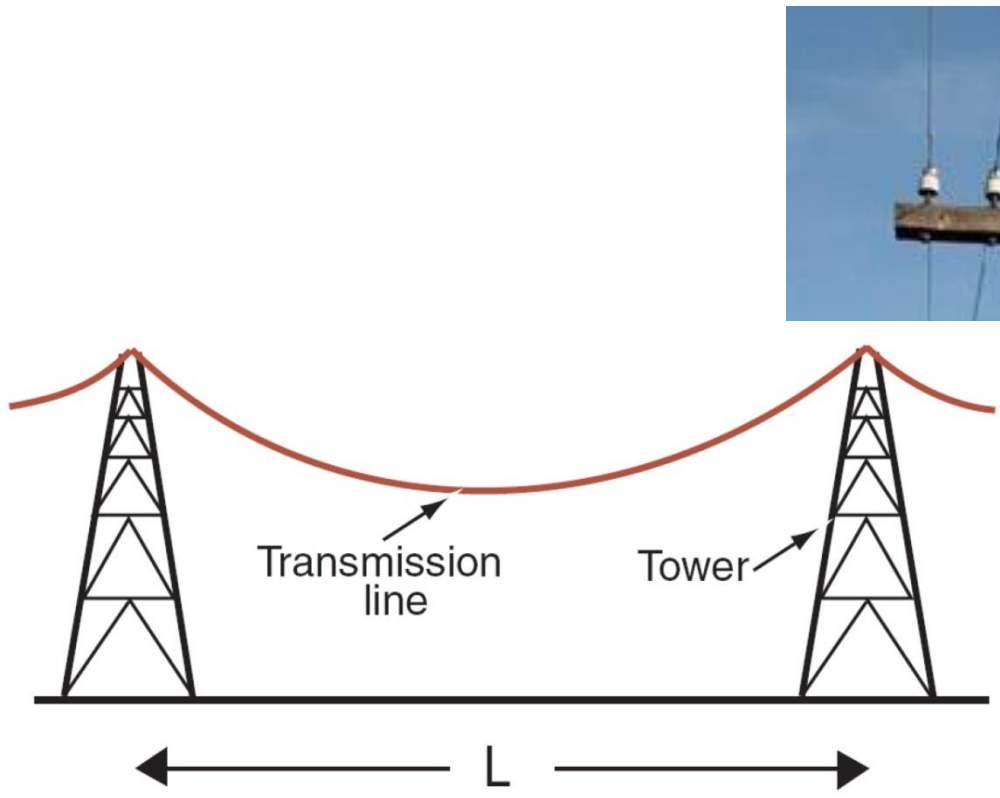




**Case Study 16:
Materials for
Long Span Transmission line**



Objective	<ul style="list-style-type: none">• Maximize current flux
Constraints	<ul style="list-style-type: none">• Easy to manufacture• Must be strong?• Dimensions fixed
Free Variables	<ul style="list-style-type: none">• Choice of material





Index from General Knowledge

Case Study 16: Materials for Long Span Transmission line

Wiedemann-Franz Law

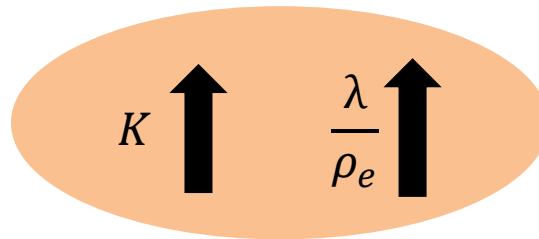
$$L \propto K = \frac{\lambda}{\rho_e}$$

L Lorenz number

K Electrical conductivity

λ Thermal conductivity

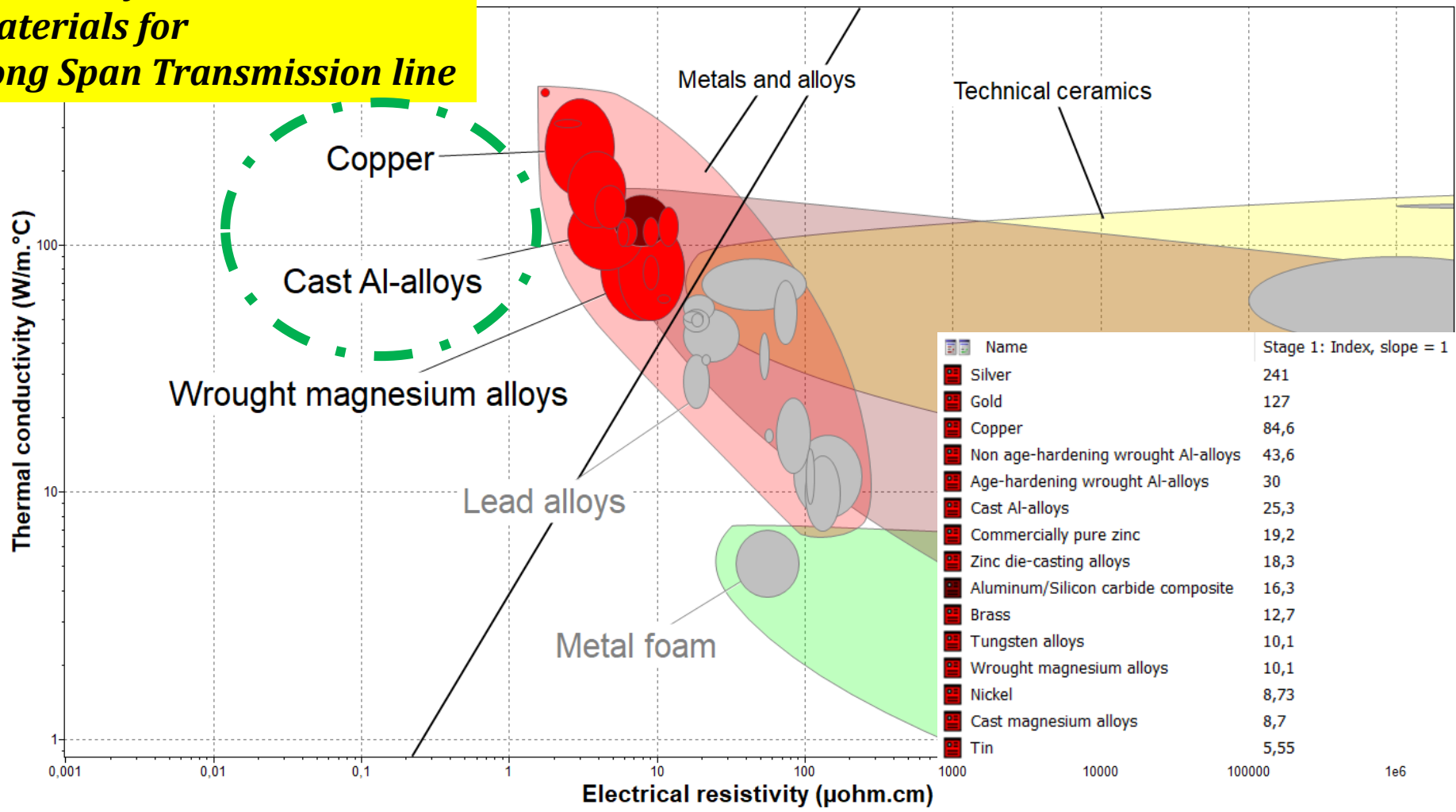
ρ_e electrical resistivity



[<http://hyperphysics.phy-astr.gsu.edu/hbase/thermo/thercond.html>]



**Case Study 16:
Materials for
Long Span Transmission line**





How the shape can change the properties → Decrease the I

Case Study 16: Parenthesis about real life

On a standard shape

$$n\pi r^2 = b^2 = A$$

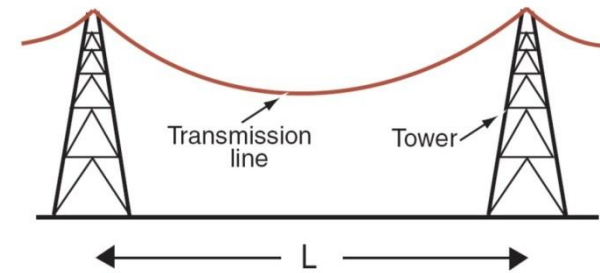
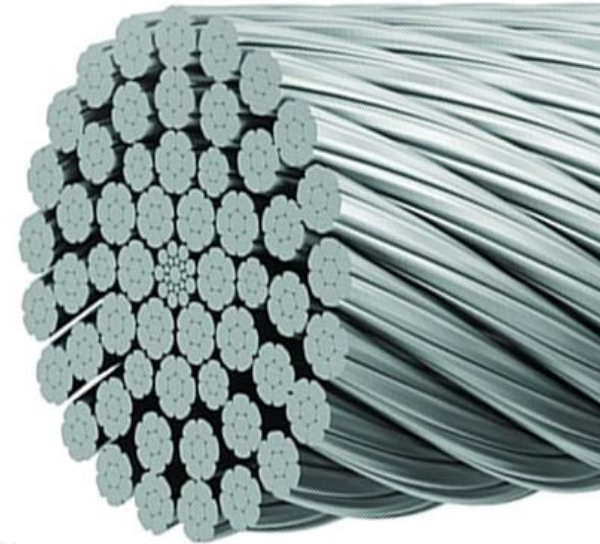
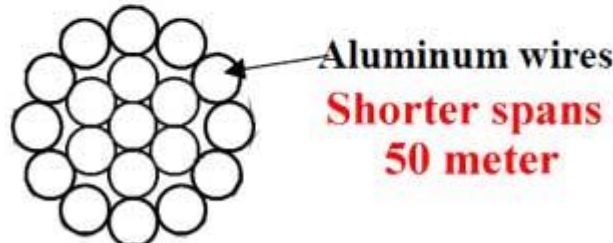
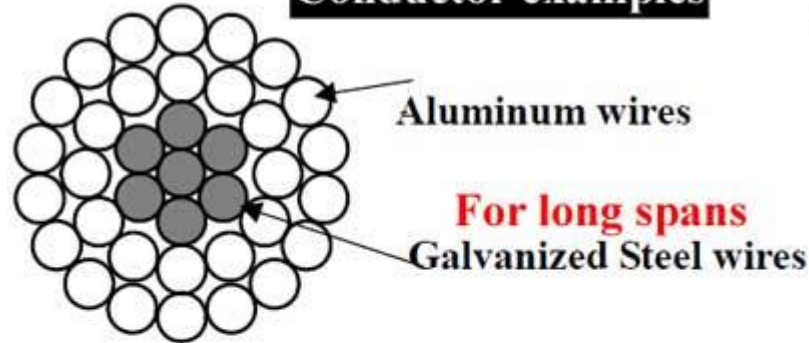
$$I_0 = \frac{A^2}{12} = \frac{(n\pi r^2)^2}{12}$$

On the single wire

$$I = I_0 \cdot n = \frac{n\pi r^4}{4}$$

$$\frac{I}{I_0} = \frac{\frac{n\pi r^4}{4}}{\frac{(n\pi r^2)^2}{12}} = \frac{3}{n\pi} \propto \frac{1}{n}$$

Conductor examples

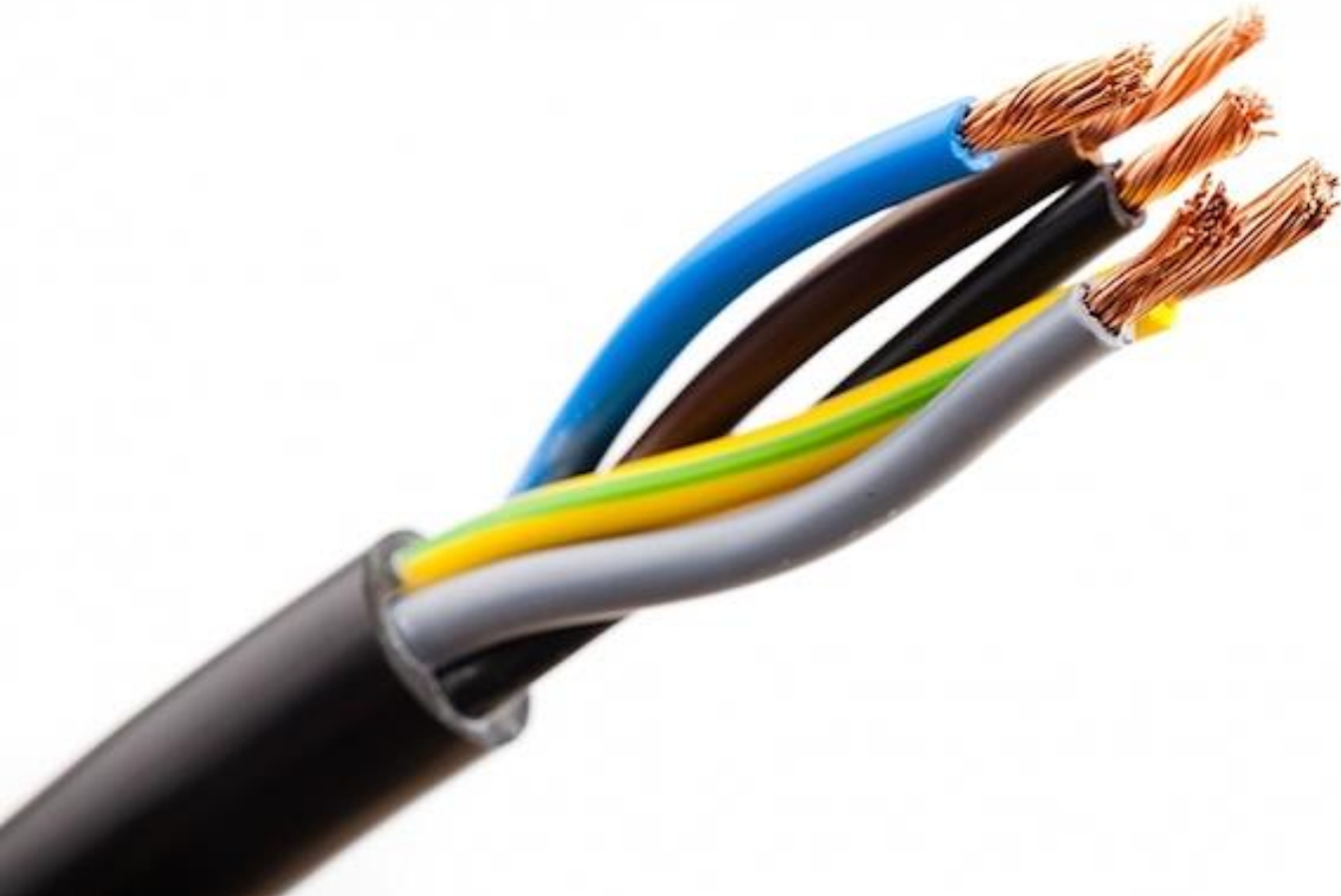




**Case Study 17:
Materials for Light Cable**



Objective	<ul style="list-style-type: none">• Minimize the mass
Constraints	<ul style="list-style-type: none">• Good conductor• Easy to manufacture• Dimensions fixed
Free Variables	<ul style="list-style-type: none">• Choice of material





Index from « Equations »

Case Study 17: Materials for Light Cable

Pouillet's Law

$$r = \rho_e \frac{L}{A}$$



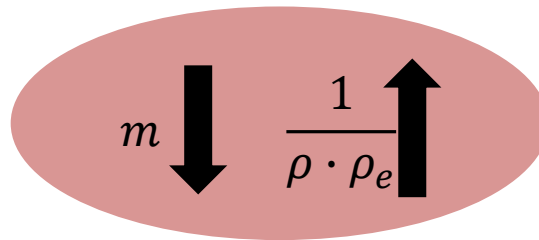
$$A = \rho_e \frac{L}{r}$$

$$m = \rho \cdot L \cdot A$$

$$m = (\rho \cdot \rho_e) \cdot \frac{L^2}{r}$$

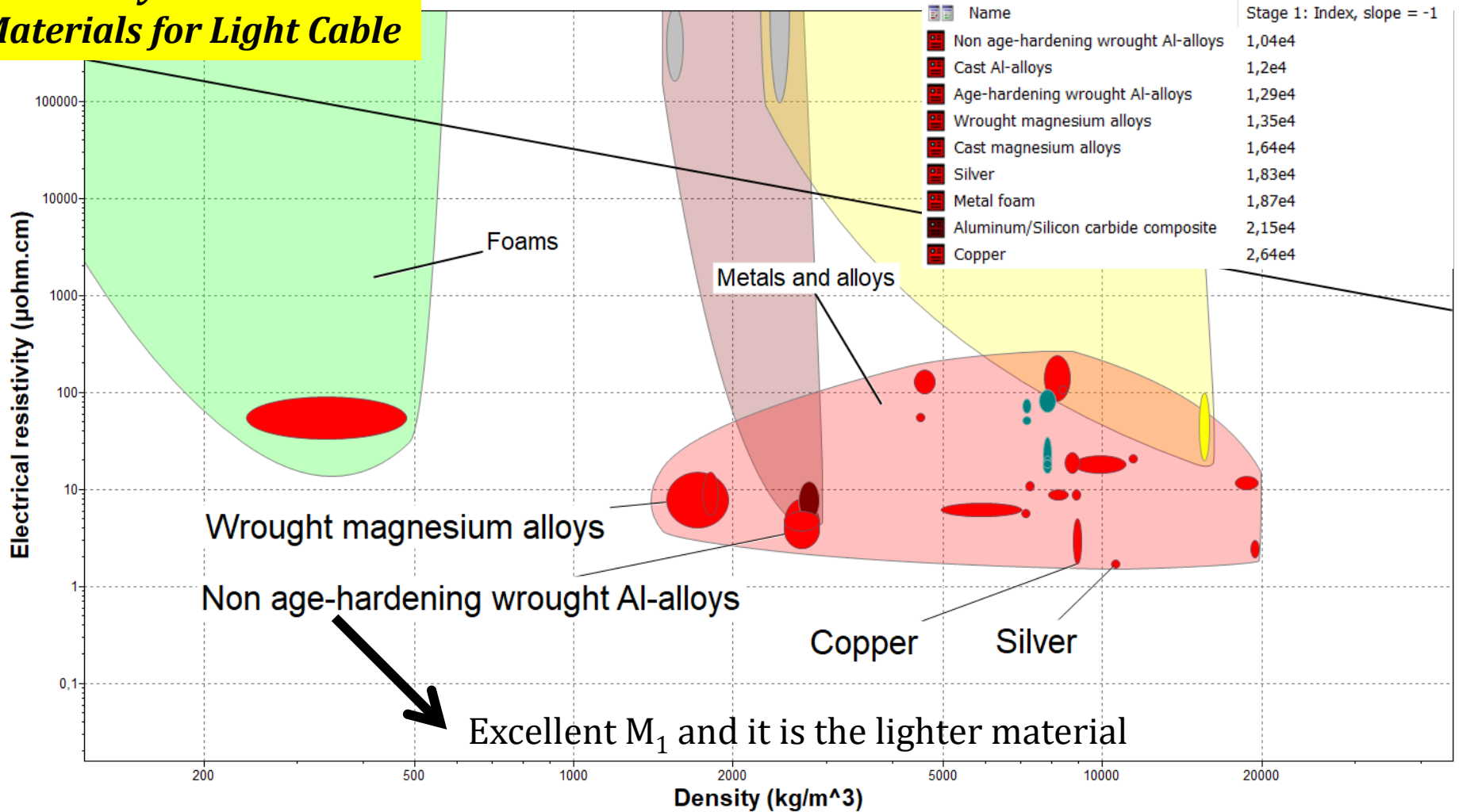
ρ_e Electrical resistivity

r Electrical resistance of a uniform specimen of the material





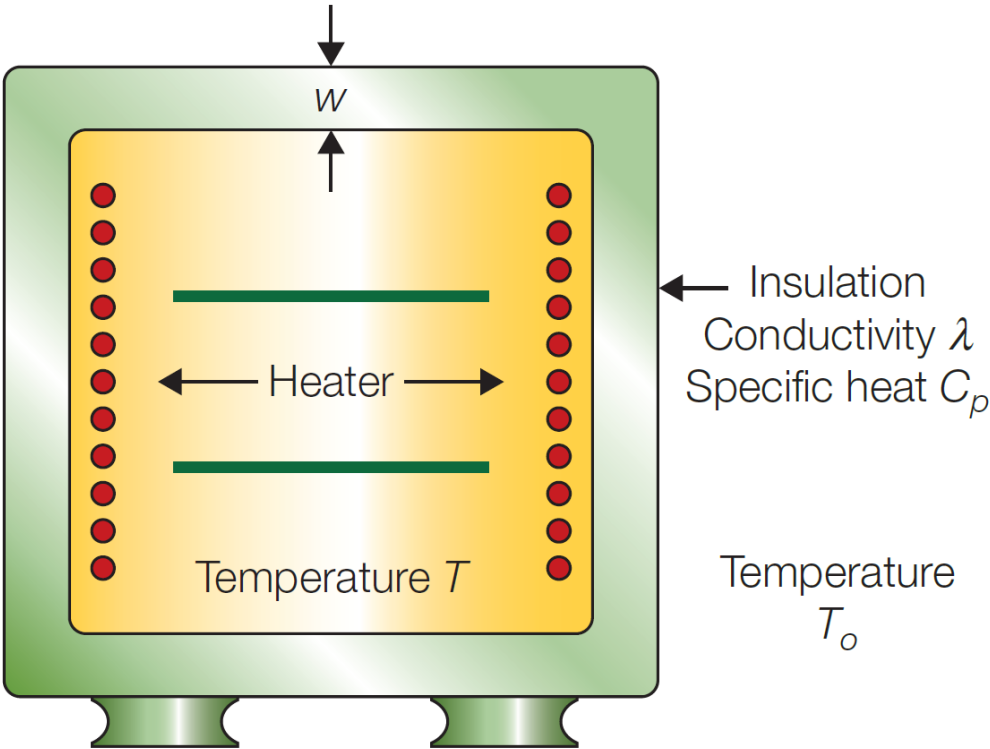
Case Study 17: Materials for Light Cable





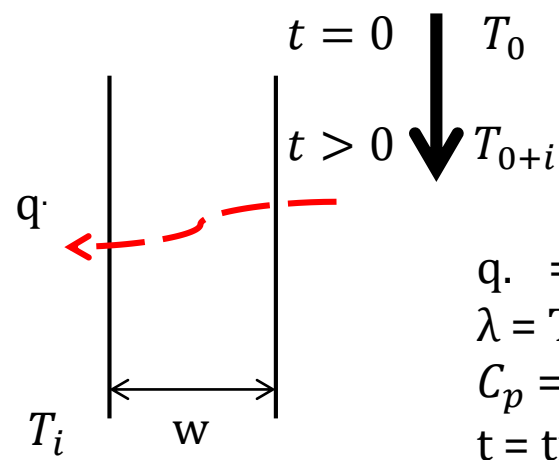
**Case Study 18:
Materials for Kiln Walls**

Objective	<ul style="list-style-type: none">Minimize energy consumed in firing cycle
Constraints	<ul style="list-style-type: none">Maximum and minimum operating temperaturePossible limit on W for space reasons
Free Variables	<ul style="list-style-type: none">Choice of material





Case Study 18: Materials for Kiln Walls



q = Heat flux
 λ = Thermal conductivity
 C_p = Specific heat capacity
 t = time

Heat loss by conduction

$$Q_1 = -\lambda \frac{(T_i - T_0)}{w} \cdot t$$

Heat (loss) absorbed by walls

$$Q_2 = \frac{\Delta T \cdot w \cdot \rho \cdot C_p}{2}$$

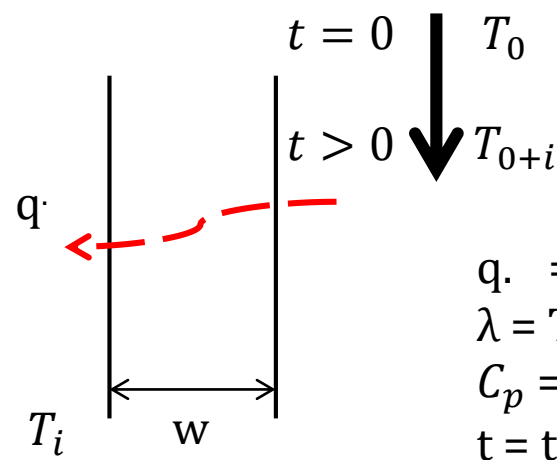
Energy-efficient kiln walls? Reduce λ and w

Total energy consumed during the use $Q = Q_1 + Q_2$

Thus, we want a Minimum $Q(w) \rightarrow$ a w that correspond to $\frac{dQ}{dw} = 0$



**Case Study 18:
Materials for Kiln Walls**



q = Heat flux
 λ = Thermal conductivity
 C_p = Specific heat capacity
 t = time

a = Thermal diffusivity

$$Q = Q_1 + Q_2 = -\lambda \frac{(T_i - T_0)}{w} \cdot t + \frac{\Delta T \cdot w \cdot \rho \cdot C_p}{2}$$

For an optimum thickness

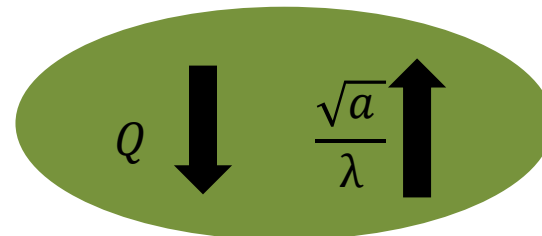
$$\frac{dQ}{dw} = 0 = -\lambda \frac{(T_i - T_0)}{w^2} \cdot t + \frac{\Delta T \cdot w \cdot \rho \cdot C_p}{2}$$

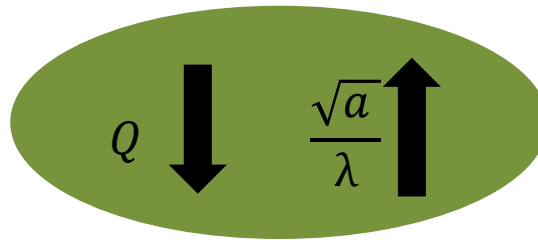
$$w^* = \sqrt{\frac{2\lambda t}{\rho \cdot C_p}} = \sqrt{2at}$$

So, if we consider

$$Q = Q_1 + Q_2 \longrightarrow Q = \frac{\Delta T \cdot \sqrt{2t} \cdot \lambda}{\sqrt{a}}$$

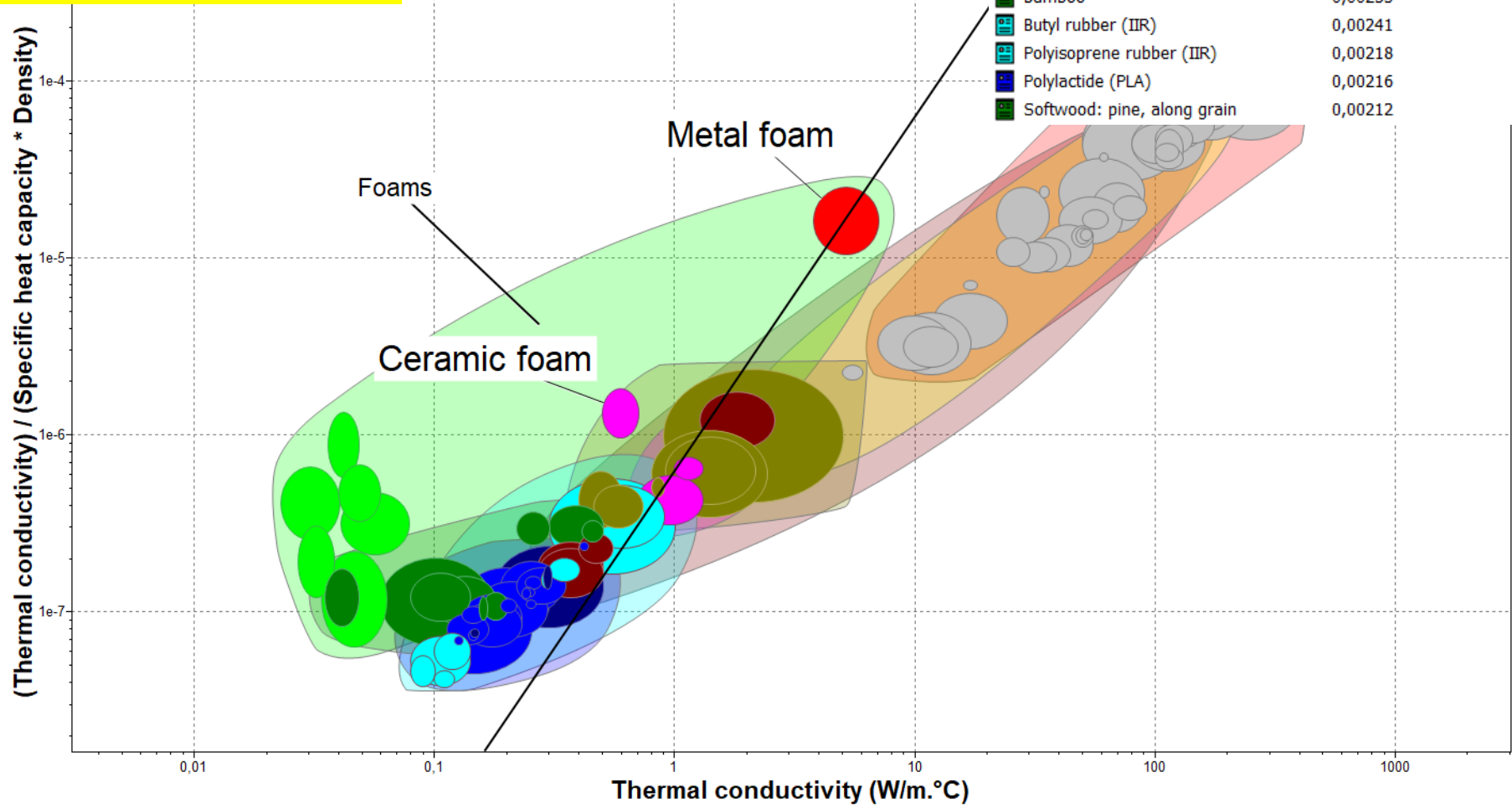
$$w^* = \sqrt{2at}$$

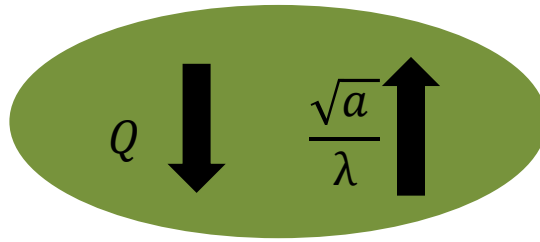




**Case Study 18:
Materials for Kiln Walls**

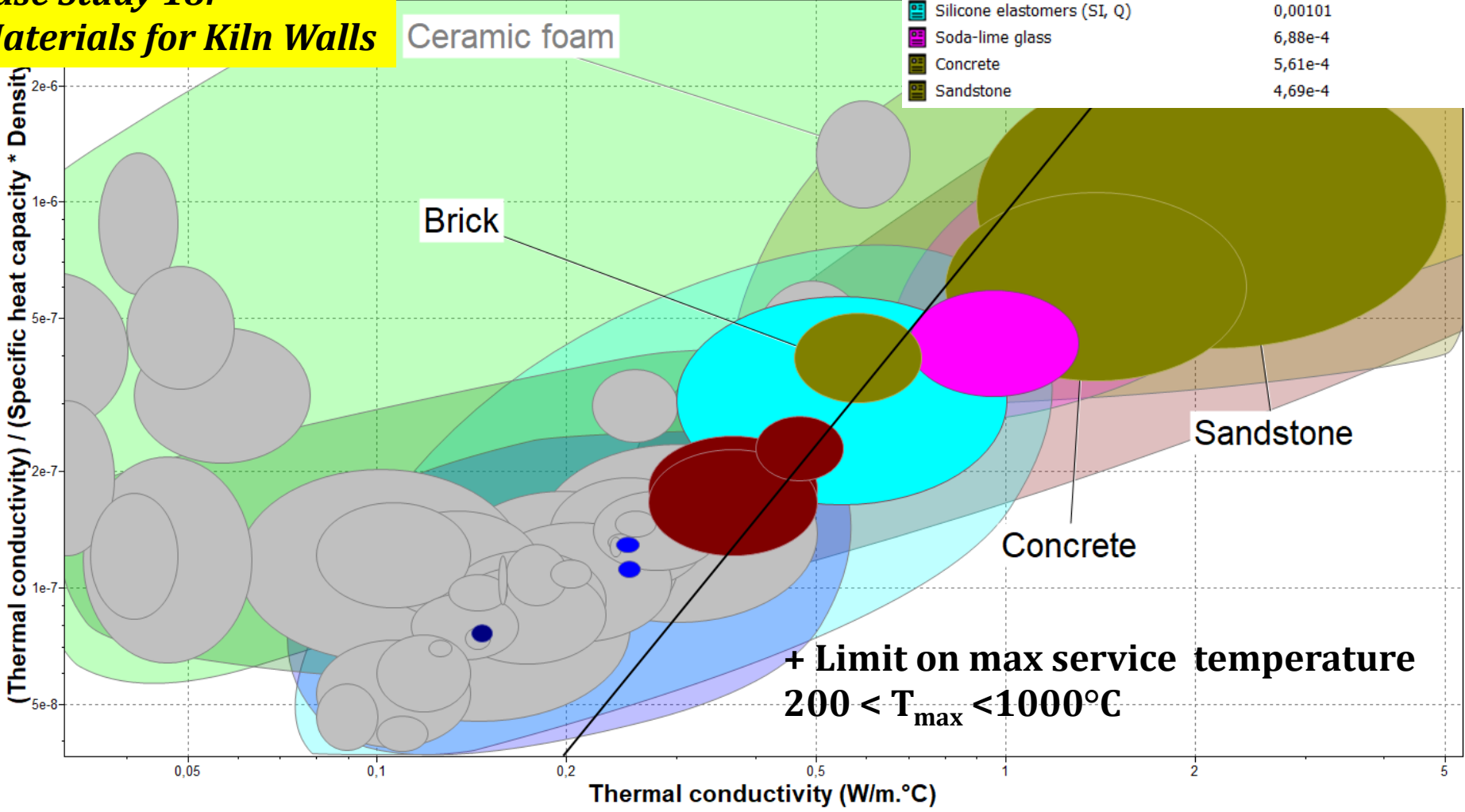
Name	Stage 1: Index, slope = 2
Flexible Polymer Foam (VLD)	0,0226
Rigid Polymer Foam (LD)	0,0212
Flexible Polymer Foam (LD)	0,0142
Rigid Polymer Foam (MD)	0,0137
Flexible Polymer Foam (MD)	0,00996
Cork	0,00848
Rigid Polymer Foam (HD)	0,00744
Paper and cardboard	0,00335
Softwood: pine, across grain	0,0033
Bamboo	0,00253
Butyl rubber (IIR)	0,00241
Polyisoprene rubber (IIR)	0,00218
Poly lactide (PLA)	0,00216
Softwood: pine, along grain	0,00212





**Case Study 18:
Materials for Kiln Walls**

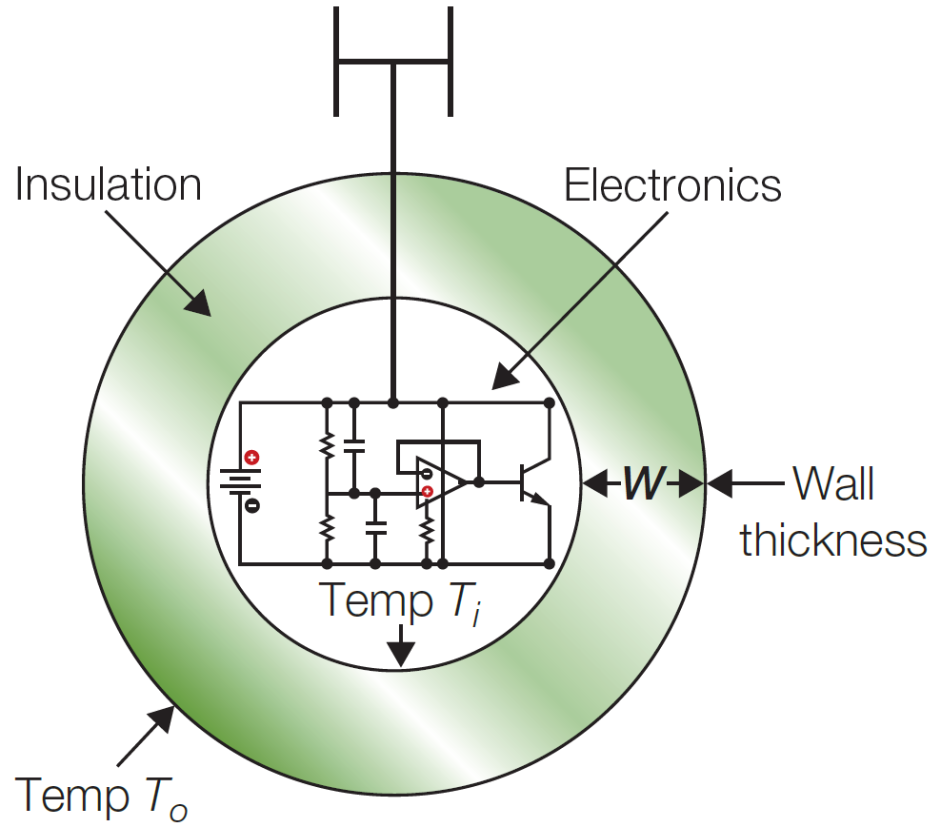
Name	Stage 1: Index, slope = 2
Phenolics	0,00189
Polyetheretherketone (PEEK)	0,00144
Polytetrafluoroethylene (Teflon, PTFE)	0,00133
Sheet molding compound, SMC, polyester ...	0,00116
Dough (Bulk) molding compound, DMC (BM...	0,00111
Brick	0,00109
GFRP, epoxy matrix (isotropic)	0,00102
Silicone elastomers (SI, Q)	0,00101
Soda-lime glass	6,88e-4
Concrete	5,61e-4
Sandstone	4,69e-4





Case Study 19:
Materials for Insulation for
Short-Term isothermal
containers

Objective	<ul style="list-style-type: none">Maximize the time t before internal temperature changes when external temperature suddenly drops
Constraints	<ul style="list-style-type: none">Wall thickness (W)
Free Variables	<ul style="list-style-type: none">Choice of the material





**Case Study 19:
Materials for Insulation for
Short-Term isothermal
containers**

Fick's Law for the temperature
at Steady state

$$q = -\lambda \frac{(T_i - T_0)}{w}$$

Fick's Law for the temperature
at not-steady state

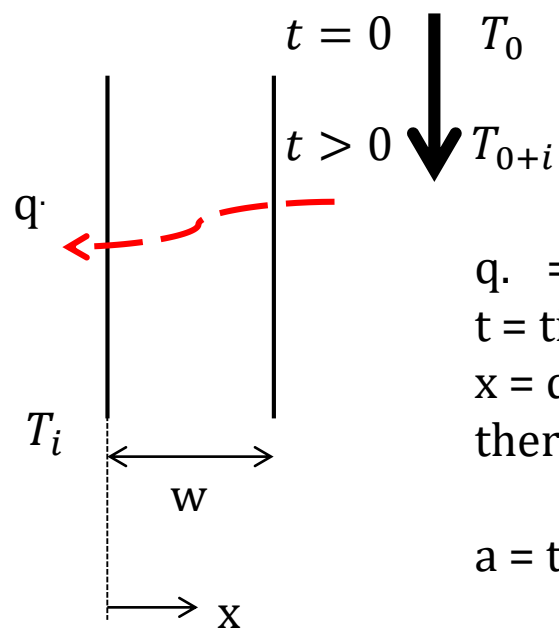
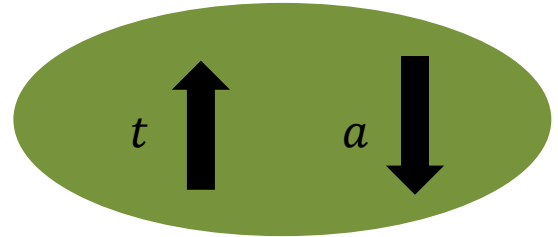
$$\frac{\partial T}{\partial t} = a \frac{\partial^2 T}{\partial x^2}$$

Too complex, so
we use

$$a = \frac{\lambda}{\rho \cdot C_p}$$

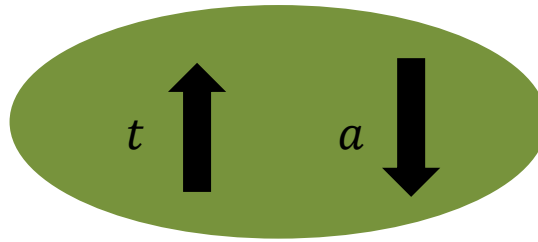
$$x \cong \sqrt{2at} \leq w$$

$$t = \frac{w^2}{2a}$$



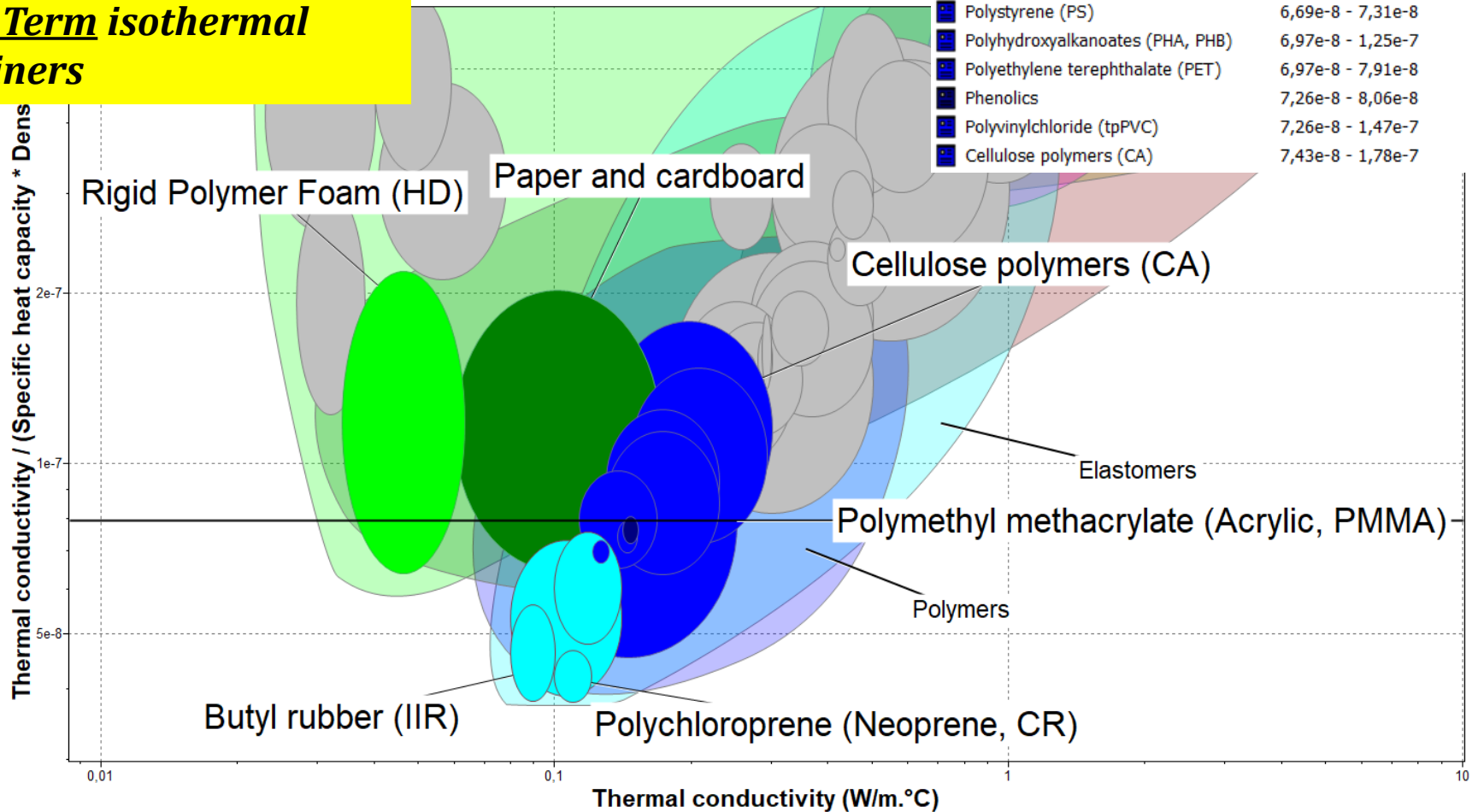
q = Heat flux
 t = time
 x = distance of propagation of
thermal equilibrium
 a = thermal diffusivity

**WE ARE NOT MINIMIZING THE HEAT FLUX!!!
WE WANT TO MAXIMIZE THE TIME t
BEFORE T_0 CHANGE CONSIDERABLY**



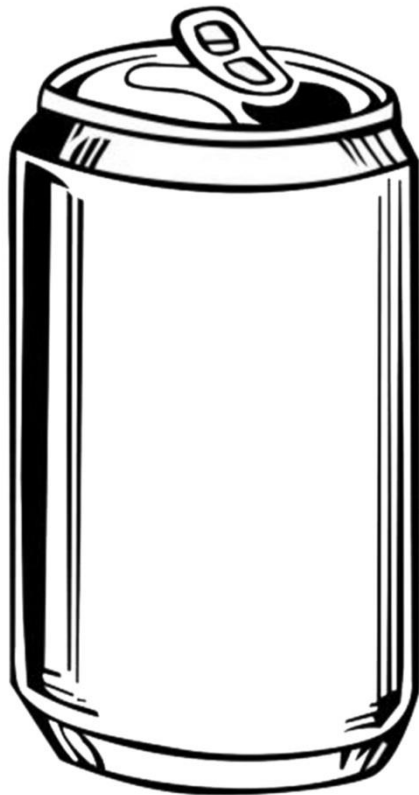
**Case Study 19:
Materials for Insulation for
Short-Term isothermal
containers**

Name	Thermal conductivity / (Sp...
Polychloroprene (Neoprene, CR)	3,8e-8 - 4,66e-8
Butyl rubber (IIR)	3,82e-8 - 5,63e-8
Polyisoprene rubber (IIR)	3,9e-8 - 7,3e-8
Polymethyl methacrylate (Acrylic, PM...	4,55e-8 - 1,37e-7
Natural rubber (NR)	4,8e-8 - 7,57e-8
Starch-based thermoplastics (TPS)	6,38e-8 - 1,14e-7
Rigid Polymer Foam (HD)	6,42e-8 - 2,19e-7
Paper and cardboard	6,51e-8 - 2,02e-7
Polypropylene (PP)	6,56e-8 - 9,72e-8
Polystyrene (PS)	6,69e-8 - 7,31e-8
Polyhydroxyalkanoates (PHA, PHB)	6,97e-8 - 1,25e-7
Polyethylene terephthalate (PET)	6,97e-8 - 7,91e-8
Phenolics	7,26e-8 - 8,06e-8
Polyvinylchloride (tpPVC)	7,26e-8 - 1,47e-7
Cellulose polymers (CA)	7,43e-8 - 1,78e-7





**Case Study 20:
Process for a Can**

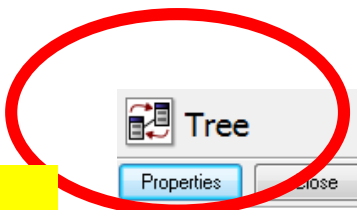


Objective	<ul style="list-style-type: none">• Find an High production + Cheap process
Constraints	<ul style="list-style-type: none">• Shape : Dished Sheet• Physical aspect :<ol style="list-style-type: none">1. Mass range max 1 kg2. Tolerance 0,2-0,5 mm• Primary Process
Free Variables	<ul style="list-style-type: none">• Choice of process





**Case Study 20:
Process for a Can**



Link Record Number Passed

MaterialUniverse: \ Metals and alloys 92

Tree Stage X

Stage Title:

[MaterialUniverse:\Metals and alloys]

Trees

ProcessUniverse

- ProcessUniverse
- Shaping

Preview

Choose records from the ProcessUniverse tree. The chosen records will pass the selection.



Dished sheet, Mass range, Tolerance, Secondary shaping processes

Properties

Apply

Clear

Click on the headings to show/hide selection criteria

▼ Shape

- Circular prismatic
- Non-circular prismatic
- Flat sheet
- Dished sheet
- Solid 3-D
- Hollow 3-D

▼ Physical attributes

	Minimum	Maximum	
Mass range	<input type="text"/>	1	kg
Range of section thickness	<input type="text"/>	<input type="text"/>	mm
Tolerance	0,2	0,5	mm
Roughness	<input type="text"/>	<input type="text"/>	μm
Cutting speed	<input type="text"/>	<input type="text"/>	m/s
Minimum cut width	<input type="text"/>	<input type="text"/>	mm

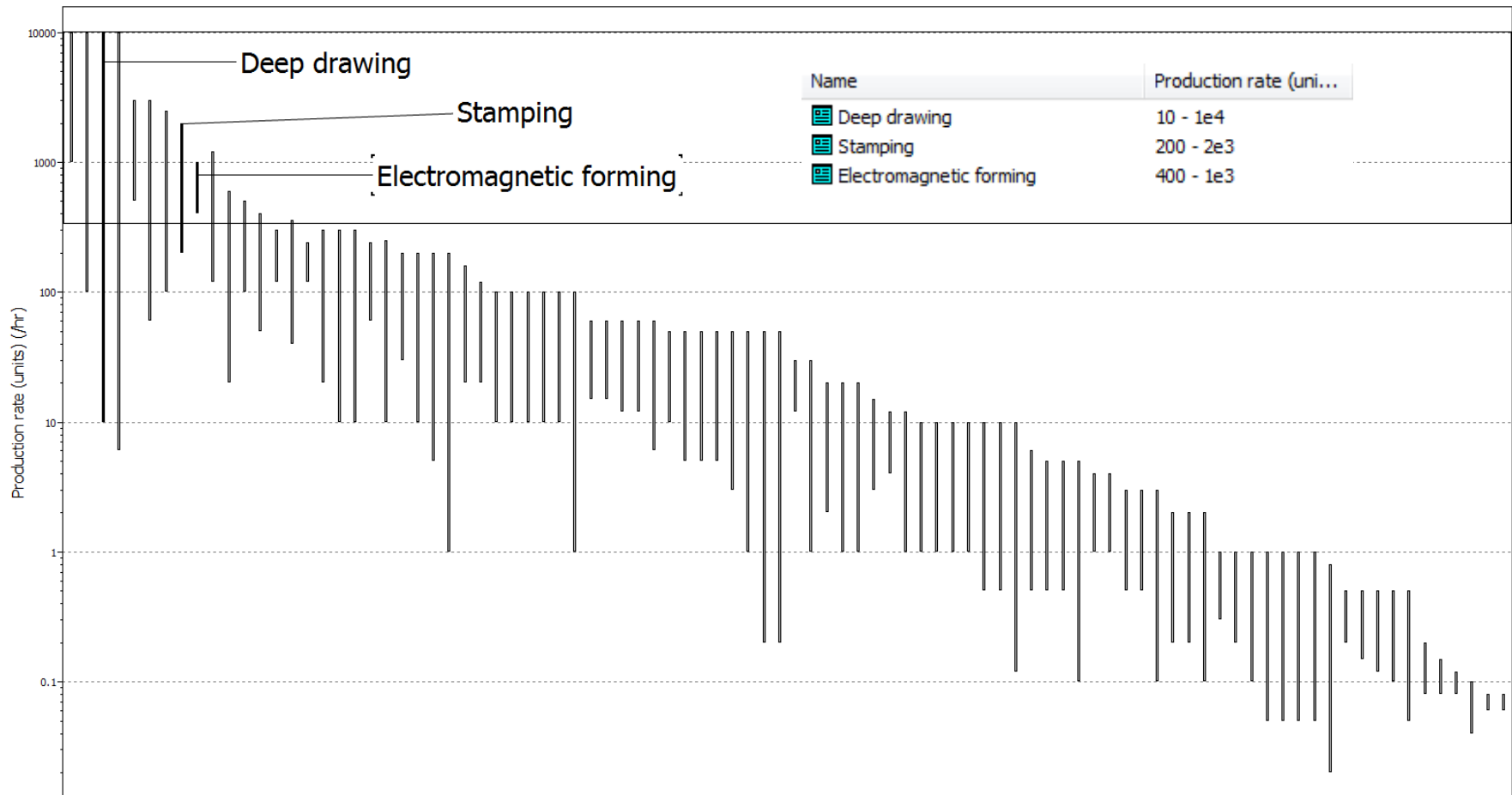
▼ Process characteristics

- Primary shaping processes
- Secondary shaping processes
- Machining processes
- Cutting processes
- Prototyping
- Discrete
- Continuous

Case Study 20: Process for a Can



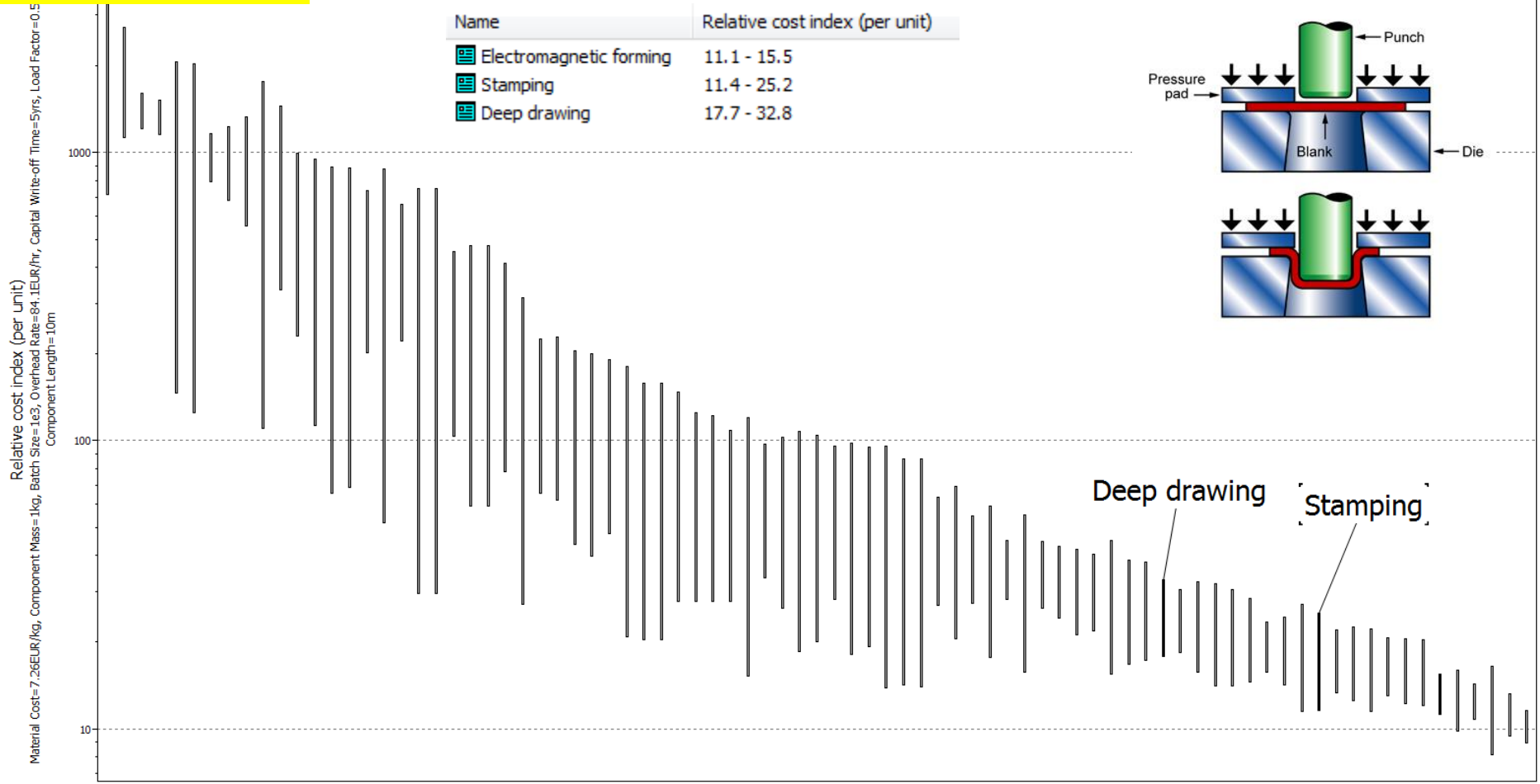
Case Study 20: Process for a Can





Results : Deep drawing (deep drawing is a particular stamping)

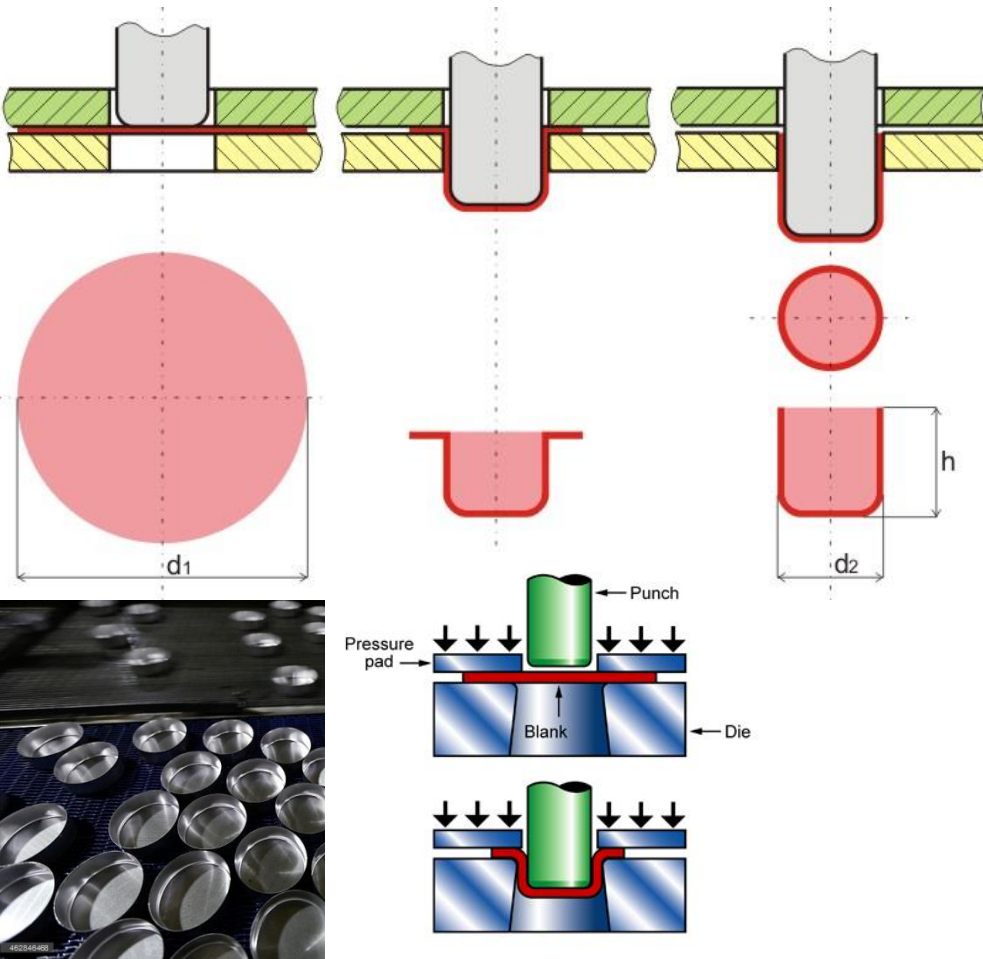
Case Study 20: Process for a Can





**Case Study 20:
Process for a Can**

Deep drawing (deep drawing is a particular stamping)





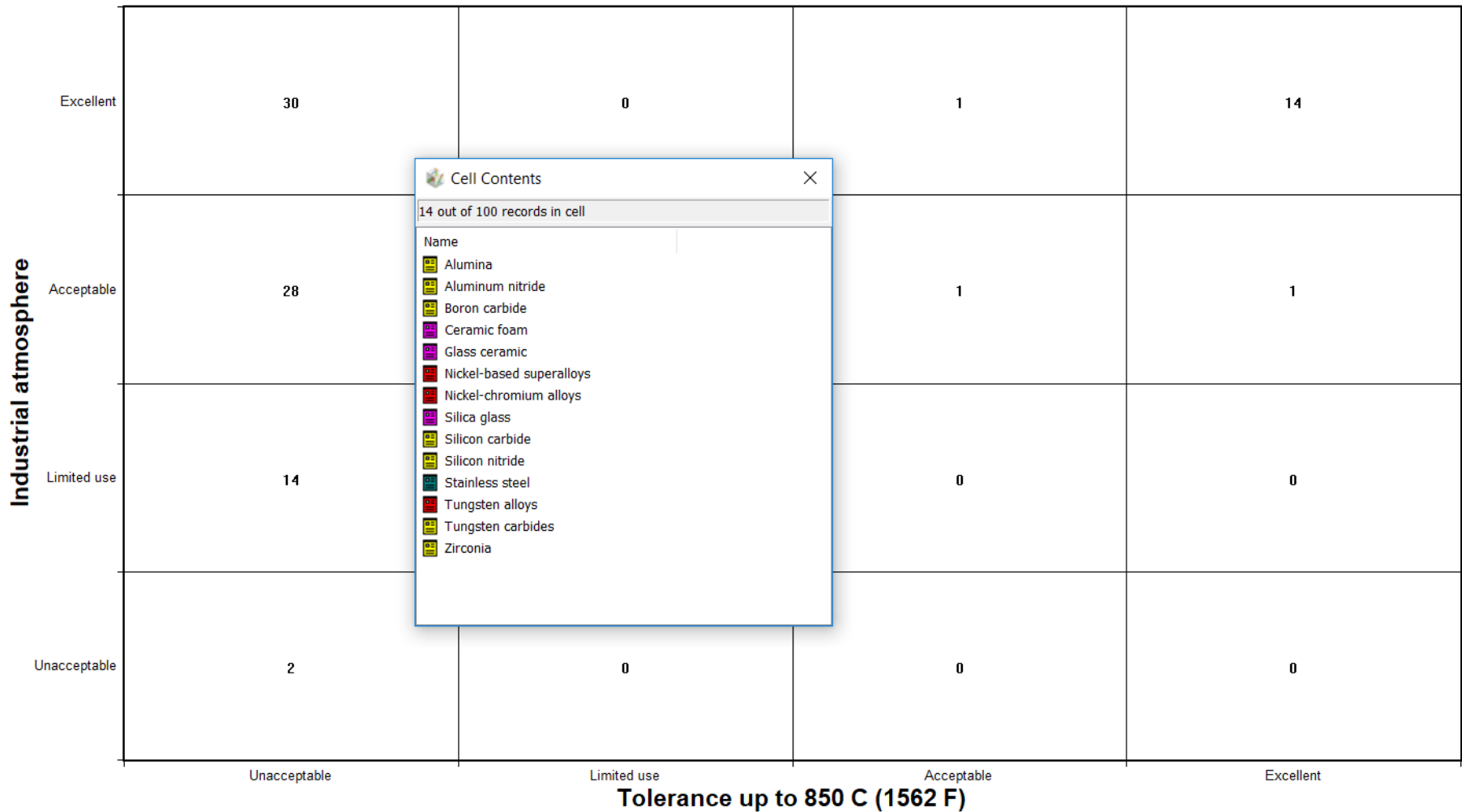
Secret slides : Kubota case

Kubota





Secret slides : Kubota case





Secret slides : Kubota case

